# AIRFOIL BOUNDARY LAYER CALCULATIONS USING INTERACTIVE METHOD AND e ${ }^{\mathrm{n}}$ TRANSITION PREDICTION TECHNIQUE 

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Boundary layer calculations are performed around an airfoil and its wake. Smith-van Ingen transition prediction method is employed to find the transition from laminar to turbulent flow. First, potential flow around the airfoil is solved with the Hess-Smith panel method. The resulting velocity distribution is input to the boundary layer equations in order to find a so called blowing velocity distribution. The output of the boundary layer equations are also used to compute the location of onset of transition using the Smith-van Ingen $e^{n}$ transition prediction method. The obtained blowing velocity distribution is fed back to the panel method to find a velocity distribution which includes the effects of viscosity. The procedure described is repeated until convergence is observed. A computer program is developed using the theory. Results obtained are in good accord with measurements.

[^0]
# KANAT PROFILİ SINIR TABAKASININ ETKİLEŞİMLİ METOD VE en GEÇİŞ TAHMİN TEKNİĞİ KULLANILARAK HESAPLANMASI 

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Kanat profili ve ard iz bölgesinde sınır tabakası hesaplamaları yapılmıştır. Geçiş tahmini için Smith-van Ingen geçiş tahmin metodu kullanılmıştrr. İlk önce Hess-Smith panel metodu kullanılarak kanat profili etrafındaki potansiyel akım çözülmüş ve hız dağılımı hesaplanmıştır. Elde edilen hız dağılımı ile sınır tabakası denklemleri çözülmüş ve yüzey üfleme hızları bulunmuştur. Sınır tabakası denklemlerinin sonuçları, Smith-van Ingen $\mathrm{e}^{\mathrm{n}}$ metodu ile türbülansa geçiş noktasının belirlenmesi için kullanılmıştır. Elde edilen yüzey üfleme hızları potansiyel akım çözümünde sınır şartları olarak kullanılmış ve sınır tabakasının etkilerini de içeren potansiyel akım bulunmuştur. Bu yöntem, yakınsama gözlemleninceye değin sürdürülmüştür. Söz konusu teori kullanılarak bir bilgisayar programı yazılmış ve iki adet kanat profili etrafinda denenmiştir. Sonuçlar deneysel ölçümler ile uyumludur.

Anahtar Kelimeler : Sıınır Tabaka Akışı, Geçiş Tahmini, Yüzey Üfleme Hızı, Kanat Profili Performansı, Etkileşimli Yöntem

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To my family, without whom this study would be impossible...

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## CHAPTER1

## INTRODUCTION

As lifting sections play an essential role in performance of any aerodynamic system, two questions are of particular importance. First is the problem of designing lifting sections, where the second is to analyze performance characteristics. Analysis of lifting sections is crucial from design of an airfoil to design of an aircraft.

In order to analyze a lifting section, mainly two approaches are applicable. One is to solve either full Navier-Stokes (N-S) equations or to solve them in a simplified manner, which is called the thin-layer Navier-Stokes equations, where diffusion terms in the mean flow direction are neglected. The main complexity of the so called thin layer Navier-Stokes equations arise from the fact that, the whole flow field has to be treated simultaneously, since these equations are not parabolic in nature and therefore does not allow one to use techniques such as marching technique. Obviously, since one has to handle the whole flow field at once, this method requires some large amount of computational power and storage space, which makes it unfeasible.

An alternative method is the so called Interactive Boundary Layer method. In this method, the major assumption is the one suggested by Prandtl, which implies that for sufficiently high Reynolds numbers, flow over a lifting section can be considered as superposition of two different layers, a thin boundary layer, or viscous layer, where all the viscous effects are included, and an inviscid outer layer. As stated in [1], it is assumed that the thickness of the boundary layer is very small compared to the characteristic length scale of the flow $(\delta \ll L)$. Using this assumption, Navier-Stokes equations can be further simplified. Adding the assumptions that diffusion in the mean flow direction, normal pressure gradient are neglected and also convection terms in the normal direction are negligible compared to convection terms in the mean flow direction; one can obtain a resultant flow in which pressure across the boundary layer can be considered constant. The advantage that these equations bring is their parabolic nature, which enables one to solve them using the so called marching technique.

In order to keep an uncoupled flow, thermal effects are neglected which yields the fact that variations in fluid properties like density and viscosity are not accounted for. Such flows can be expected in incompressible flows where the Mach number less than a predetermined value of 0.3 in order to keep the errors due to this assumption less than $10 \%(M \leq 0.3)$, and of course an adiabatic wall, where the wall of the lifting section does not allow any thermal flux to flow.

Inclusion of the wake region enables one to conform to the Kutta condition in the wake, when it is not possible to satisfy it at the trailing edge at high angles-of-attack mainly due to a separation at either one of the surfaces. Also as noted in references $[2,3,4]$ when the wake is not taken into account, the coefficient of lift, $C_{L}$, and the stall angle, $\alpha_{\text {stall }}$, are overestimated.

In order not to overestimate the section lift coefficient and the stall angle-of-attack, the blowing velocity distribution has to be calculated precisely, as the above explained overestimation is due to underestimation of the blowing velocity distribution as cited in [1].

The blowing velocity itself, creates the dividing streamline which approximately describes the edge of the boundary layer and starts from the stagnation point proceeding in the downstream direction. The blowing velocity is defined by the below formula:

$$
\begin{equation*}
V_{n}=\frac{d}{d \xi}\left(U_{e} \delta^{*}\right) \tag{1.1}
\end{equation*}
$$

Where $U_{e}$ is the edge velocity, $\xi$ is the non-dimensional surface distance, and $\delta^{*}$ is the displacement thickness.

In addition to work done by Özgen in 1994 as a Master's Thesis study [1], in which wake region was added to the interactive boundary layer solver enabling it to result in rational solutions at high angles-of-attack; this work is aimed to introduce a useful method for calculating the laminar to turbulent transition point for the above explained solver, so that transition prediction is based on methods that have stronger physical foundations and to standardize this procedure.

As this thesis is a continuation to the Master's Thesis of Özgen [1], chapters 2, 3 and 5, and appendices A and B are taken form reference [1].

The methodology used in this study may be summarized as follows: First, the inviscid flow equations are solved in order to obtain an edge-velocity distribution throughout the chord-wise stations of a lifting section, which are then input into a boundary layer solver. The two solvers, namely the potential and boundary layer solvers, are communicated via the so called blowing velocity distribution. Each communication of the potential and the boundary layer solver is called a cycle. After certain number of cycles, as the results converge to some values, with the input from the user that the flow field is fully laminar, the boundary layer solver is modified to output the values of the non-dimensional velocity and some of its derivatives as required by the solution of the Orr-Sommerfeld equation. The $e^{n}$ technique is based on solution of the Orr-Sommerfeld (O-S) equation and is explained in detail in Chapter 4. After the solution of the O-S equation is obtained, one may choose to obtain the neutral stability curve, or to carry on to find amplification rates, which constructs the basis for Smith-van Ingen $\mathrm{e}^{\mathrm{n}}$ transition prediction technique. After the transition is found, a new boundary layer solution is then obtained with the value of transition location given externally.


Figure 1.1 Brief Flowchart of the Solution Procedure

Aside from correlation methods, for predicting transition, the available methods can be listed as $e^{n}$ method which is based on solution of linear stability equations, another method based on parabolized stability equations (PSE), and finally Direct Navier Stokes (DNS) method based on the solution of unsteady Navier-Stokes equations.

As DNS method has a potential, its requirement of huge computational power which yields in large computing times, and large storage brings the fact that this method requires time in order to achieve such high computational power, and therefore it will not become a standard for transition prediction in the near future. As noted by Cebeci and Cousteix in reference [5], another approach is the parabolized stability equations method, which is still under development, and hence is not mature yet, and therefore the only way of solution available immediately is the Smith-van Ingen $e^{\mathrm{n}}$ method, which is based on the solutions of the linear stability equations.

In two-dimensional incompressible flows, the linear stability equations are embedded into a single $4^{\text {th }}$ order ordinary differential equation called the Orr-Sommerfeld equation which is given below:
$\left(\phi^{i v}-2 \alpha^{2} \phi^{\prime \prime}+\alpha^{4} \phi\right)=i R\left[(\alpha u-\omega)\left(\phi^{\prime \prime}-\alpha^{2} \phi\right)-\alpha u^{\prime \prime} \phi\right]$

In the above formulation, primes stand for derivation with respect to $y, u$ denotes the velocity profile in the main stream direction, and $\phi$ is the complex amplitude of the stream function, therefore consists of a real and an imaginary part. $\alpha$ is the wave number of the disturbance, and $\omega$ stands for the circular frequency. Finally, $v$ is the kinematic viscosity of the Newtonian fluid, in which the flow is investigated.

There are two possibilities to solve for the Orr-Sommerfeld equation. In the first case, $\alpha$ is real and $\omega$ is complex, where the amplitude changes with time according to the formulation $\exp \left(-\omega_{i} t\right)$ and is named as the temporal amplification theory. The second case is the one where $\omega$ is real but $\alpha$ is complex $\alpha \equiv\left(\alpha_{r}+i \alpha_{i}\right)$. In the latter case, amplitude of the disturbance varies with distance as $\exp \left(-\alpha_{i} t\right)$, and is called the spatial amplification theory.

The mean flow is assumed to be steady and spatial amplification theory is used, since it represents the disturbance growth in a steady-boundary layer more faithfully. Hence, in spatial amplification theory, the amplification rate and amplitude of a disturbance wave is independent of time but is a function of the distance from the stagnation point measured in downstream direction, the change of amplitude can be analyzed in a point by point manner.

## CHAPTER 2

## BOUNDARY LAYER EQUATIONS FOR 2D FLOWS

Although integral methods such a Pohlhausen method or Thwaites method are still used for calculating initial laminar regions up to the onset of transition, they are only good for quick and rough estimates for a restricted class of laminar flows and their popularity in the pre-computer era have ended due to recent development of high speed computers as discussed in ref [5]. As differential methods are more general and accurate, such a method is preferred for this study.

In a differential method, continuity and momentum equations, and their corresponding boundary conditions have to be solved in partial differential equation form, with an accurate term for the Reynolds shear stress.

### 2.1 Continuity Equation

The compressible continuity equation is given below as:
$\frac{D \rho}{D t}+\rho \nabla \cdot V=0$

One shall note that in the above equation the capital letter D denotes total derivative. Bearing in mind that incompressible flow cases are investigated, in this study one can assume that if the fluid density remains unchanged, the continuity equation can be simplified to the below format. In the below expression, $\rho$ and $t$ denotes the density and time, where $V$ represents the velocity field with $\nabla$ being the divergence operator.
$\frac{D \rho}{D t}=0 \Rightarrow \nabla \cdot V=0$

This can also be written in Cartesian coordinates as:
$\frac{\partial u}{\partial x}+\frac{\partial v}{\partial y}+\frac{\partial w}{\partial z}=0$

In the above formulation; $u, v$ and $w$ stand for velocity components in $x, y$ and $z$ directions respectively.

### 2.2 Momentum Equation

The general momentum equation for constant density and viscosity flow is given below. Note that in the below equation, $f$ represents the net body force per unit mass exerted on the fluid as defined in reference [6].
$\rho \frac{D \vec{V}}{D t}=\rho \vec{f}-\vec{\nabla} p+\mu \nabla^{2} \vec{V}$

Taking the curl of the above equation, one obtains the vorticity transport equation with the additional knowledge: $\vec{\omega}=\vec{\nabla} \times \vec{V}$. One has to note that the term $\omega$ is the vorticity transport equation, but not the frequency term which will be introduced later in chapter 4.
$\frac{D \vec{\omega}}{D t}=\vec{\omega} \cdot \vec{\nabla} \vec{V}+\nu \nabla^{2} \vec{\omega}+\vec{\nabla} \times \vec{f}$

In equation 2.4, term in the left hand side and the first term in the right hand side stands for vortex stretching, while the second is for viscous diffusion and the last one represents body forces. Note that in a two-dimensional flow, the vorticity has no components in x and $y$ directions, but only in $z$-direction, which results in the vortex stretching term to disappear so that the above equation becomes;

$$
\begin{equation*}
\frac{D \vec{\omega}}{D t}=\nu \nabla^{2} \vec{\omega}+\vec{\nabla} \times \vec{f} \tag{2.5}
\end{equation*}
$$

Interpreting equation 2.5 , one may conclude that in the flow of our particular interest, which is two-dimensional, incompressible and with constant viscosity, the vorticity of fluid elements are constant in stream-wise directions except for the body forces and viscous diffusion cases.

In order to make equation 2.5 valid for turbulent flow cases, one may replace the fluid property terms and dependent variables with their unsteady versions. These are shown below in equation 2.6.
$u=\bar{u}+\hat{u}$
$v=\bar{v}+\hat{v}$
$w=\bar{w}+\hat{w}$
$\rho=\bar{\rho}+\hat{\rho}$
$p=\bar{p}+\hat{p}$

One shall note that the unsteady versions of the above variables, in other terms, the instantaneous quantities consist of a mean part denoted by bars, and a fluctuating part denoted by hats. It is important to recall that when one applies time averaging to any of the above perturbation quantities, the result is zero shown for velocity component $u$ below in equation 2.7 . Note that in the below equation, the term $T$ denotes the time segment in which the averaging is done.
$\frac{1}{T} \int_{0}^{T} \hat{u} d t=0$

Substituting these values to conservation equations given by eqn.2.2 and 2.5, one gets the below equations;

$$
\begin{equation*}
\frac{\partial \bar{u}}{\partial x}+\frac{\partial \bar{v}}{\partial y}+\frac{\partial \bar{w}}{\partial z}=0 \tag{2.8a}
\end{equation*}
$$

$\rho \frac{D \bar{u}}{D t}=-\frac{\partial \bar{p}}{\partial x}=\mu \nabla^{2} \bar{u}+\rho f_{x}-\rho \frac{\partial}{\partial x}\left(\overline{\hat{u}}^{2}\right)-\rho \frac{\partial}{\partial y}(\overline{\hat{u} \hat{v}})-\rho \frac{\partial}{\partial z}(\overline{\hat{u} \hat{w}})$
$\rho \frac{D \bar{v}}{D t}=-\frac{\partial \bar{p}}{\partial y}=\mu \nabla^{2} \bar{v}+\rho f_{y}-\rho \frac{\partial}{\partial x}(\overline{\hat{v} \hat{u}})-\rho \frac{\partial}{\partial y}\left(\overline{\hat{v}}^{2}\right)-\rho \frac{\partial}{\partial z}(\overline{\hat{v}} \hat{\hat{w}})$

$$
\begin{equation*}
\rho \frac{D \bar{w}}{D t}=-\frac{\partial \bar{p}}{\partial z}=\mu \nabla^{2} \bar{w}+\rho f_{z}-\rho \frac{\partial}{\partial x}(\overline{\hat{w} \hat{u}})-\rho \frac{\partial}{\partial y}(\overline{\hat{w} \hat{v}})-\rho \frac{\partial}{\partial z}\left(\overline{\hat{w}}^{2}\right) \tag{2.8d}
\end{equation*}
$$

Noting that the continuity equation remains the same, the momentum equations in the x , y , and the z -directions gained some extra terms, which are basically multiplications of two fluctuating velocity components. These are additional terms called the Reynolds stress terms. If a term consists of multiplication of two velocity components in the same direction, they are referred to as Reynolds normal stresses, otherwise they are named Reynolds shear stress terms.

This additional information lets one to visualize stresses caused by turbulence and hence define stress as superposition of laminar and turbulent stress components as follows:

$$
\begin{equation*}
\sigma_{i j}=\sigma_{i j}^{t}+\sigma_{i j}^{l} \tag{2.9}
\end{equation*}
$$

Where superscript $t$ denotes turbulent and superscript 1 denotes laminar. Also the turbulent stress component can be written in general form as defined below. The first term in the RHS denotes turbulent stresses, where the second is for laminar stresses.

$$
\begin{equation*}
\sigma_{i j}=\overline{-\rho \hat{u}_{i} \hat{u}_{j}}+\mu\left(\frac{\partial u_{i}}{\partial x_{j}}+\frac{\partial u_{j}}{\partial x_{j}}\right) \tag{2.10}
\end{equation*}
$$

### 2.3 Boundary Layer Equations

Navier-Stokes equations can be simplified by neglecting some of the viscous terms and yield to special simplified versions of the N-S equations such as, Thin Layer N-S equations, Parabolized N-S equations, Euler Equation of Boundary Layer Equations as derived in [7]. Essentially the neglecting process is accomplished via an order of magnitude analysis, and terms that have a lower order of magnitude are dropped to achieve a simplified form.

In the case of deriving Boundary Layer Equations, as noted in Chapter 1, the major assumption is that in both laminar and turbulent flows, the thickness of the boundary layer is very small compared to the characteristic length scale of the flow $(\delta \ll L)$. This
assumption leads to an unchanged continuity equation, disappearance of the $y$ momentum equation and to the fact that pressure is constant along the boundary layer, i.e. pressure is dependent only on the stream wise distance.

The resultant equations for a steady, two dimensional and incompressible flow, are given below. Derivations of these equations starting from Reynolds averaged Navier-Stokes equations can be found in many books including references $[8,9]$.

Continuity Equation: $\quad \frac{\partial u}{\partial x}+\frac{\partial v}{\partial y}=0$

Momentum equation in x-direction: $u \frac{\partial u}{\partial x}+v \frac{\partial u}{\partial y}=-\frac{1}{\rho} \frac{\partial p}{\partial x}+\frac{1}{\rho} \frac{\partial}{\partial y}\left(\mu \frac{\partial u}{\partial y}-\rho \overline{\hat{u} \hat{v}}\right)$

Momentum equation in y-direction: $\frac{\partial p}{\partial y}=0$

Terms at the LHS of equation 2.12 are the convection terms, where the first term at the RHS is the pressure term followed by viscous diffusion terms. Note that the last term includes the Reynolds shear stress term.

In order to simplify the last term on the RHS of equation 2.12, one can embed the division by density term into the terms inside the parenthesis to obtain:

$$
\begin{equation*}
\frac{\partial}{\partial y}\left(v \frac{\partial u}{\partial y}-\overline{\hat{u} \hat{v}}\right) \tag{2.14}
\end{equation*}
$$

And also using information provided by equation 2.13, the partial differentiation at the first term in the RHS of equation 2.12 may be replaced as an ordinary differentiation, and further may be combined with the Bernoulli's equation to yield the following relation. The interpretation of the resultant relation yields the fact that pressure is not a variable through the solution process but absorbed within the boundary conditions.
$\frac{d p}{d x}=-\rho U_{e} \frac{d U_{e}}{d x}$

Rewriting equation 2.12 including the above relations:
$u \frac{\partial u}{\partial x}+v \frac{\partial u}{\partial y}=U_{e} \frac{d U_{e}}{d x}+\frac{\partial}{\partial y}\left(v \frac{\partial u}{\partial y}-\overline{\hat{u} \hat{v}}\right)$

The last term at the RHS of the above equation can be expressed according to Boussinesque's relation, introducing the eddy viscosity concept, which implies:
$-\overline{\hat{u} \hat{v}}=\varepsilon_{m} \frac{\partial u}{\partial y}$

If we define a variable $b$ such that
$b=v+\varepsilon_{m}$

Then equation 2.16 becomes:
$u \frac{\partial u}{\partial x}+v \frac{\partial u}{\partial y}=U_{e} \frac{d U_{e}}{d x}+\frac{\partial}{\partial y}\left(b \frac{\partial u}{\partial y}\right)$

It shall be clarified that the above equation is valid both for laminar and turbulent flows through the proper values of $b$.

In case of no mass transfer across the wall, the associated boundary conditions for eqs. 2.11 and 2.13 become:

Wall B.C. $\quad u=0, v=0$ at the wall $(\equiv y=0)$

Edge B.C. $\quad u \rightarrow U_{e}(x) \quad$ as $\quad y \rightarrow \infty$

In the wake region, things get more complicated as one has no solid wall. Therefore, one has to introduce a dividing streamline where $y=0$, which enables one to distinguish the difference between upper and lower surfaces. It is of particular importance that as there exist a requirement for inclusion of the wake region at high the angles-of-attack. The reason for this is the assumption mentioned above stating that pressure is independent of
y becomes invalid since as separated regions occur. In addition to this, the boundary-layer assumption explained above also becomes invalid as the displacement thickness, $\delta^{*}$, which is explained in section 3.1 becomes as high as $5-6 \%$ of the chord length. In order to keep simplicity, the dependence of pressure in the normal direction is omitted and so that the boundary conditions used become:
$u \rightarrow U_{e}(x) \quad$ as $\quad y \rightarrow \infty$
$v=0 \quad$ at $\quad y=0$
$u \rightarrow U_{-e}(x) \quad$ as $\quad y \rightarrow-\infty$

In order to solve equation 2.19 around any lifting section, one has to divide the section into two as upper and lower surfaces, each starting from the stagnation point proceeding downstream. After solving for the boundary layer through these surfaces, the solutions are merged into the wake region and solutions along the wake region are carried for about three chord-lengths downstream.

## CHAPTER 3

## SOLUTION OF 2D BOUNDARY LAYER EQUATIONS

Among several different methods for solving differential boundary layer equations, two of them are very popular, namely the Crank-Nicholson method and the Keller's box scheme as described in [10] and they are compared briefly in reference [8]. Keller's box scheme is used in this study. The same method can be applied to three-dimensional boundary layer equations and both two and three-dimensional linear stability equations. As a result, the same method is used for solving stability equations described later in appendix A. Further in formation is available in references $[1,5,7,11]$.

Recalling the continuity and momentum equations for a two dimensional boundary layer:

$$
\begin{equation*}
\frac{\partial u}{\partial x}+\frac{\partial v}{\partial y}=0 \tag{2.11}
\end{equation*}
$$

$u \frac{\partial u}{\partial x}+v \frac{\partial u}{\partial y}=U_{e} \frac{d U_{e}}{d x}+\frac{\partial}{\partial y}\left(b \frac{\partial u}{\partial y}\right)$

The above equation that is equation 2.19 is applicable to both internal and external flows. On the other hand, in order to solve for external flows, which is essentially the case investigated throughout this study, the edge velocity distribution, $U_{e}(x)$, must be supplied. Via the Bernoulli's equation given below:
$-\frac{1}{\rho} \frac{\partial p}{\partial x}=U_{e} \frac{d U_{e}}{d x}$

If in a solution, one specifies the edge velocity distribution or equivalently the pressure distribution, the approach is defined as the standard problem, requiring that no separated regions within the solution domain exist, that is the wall shear stress shall never be zero ( $\equiv \tau_{w} \neq 0$ ). This is referred to as the singular behavior of boundary layer solutions in case a separation exists as described in [7].

In order to solve for the cases when there exists a separated region, the methodology applied to the solution of the boundary layer equations given above shall be altered. In order to treat the problem in a different manner, one does not input neither the edge velocity distribution nor the pressure distribution, but tries to solve for them also. In this case, one has to specify the displacement thickness distribution, $\delta^{*}(x)$, as a boundary condition and solve for the edge velocity distribution, which is referred to as the inverse method or inverse problem.

Obviously, as the displacement thickness distribution also being unknown, one has to find another way to overcome this new problem. The solution is reached through leaving these quantities as a part of the solution, and finding the correct, converged values in a manner where inviscid and viscous solutions, which are essentially potential and boundary layer solutions are interacted successively until convergence is achieved.

### 3.1 Interaction Mechanism between Inviscid and Viscous Solutions

There are two different approaches that may be used to assure interaction between inviscid and viscous solutions. The first one is called the weak interaction problem in which, the displacement thickness distribution is used as the interacting agent, and the second is called the strong interaction problem in which the blowing velocity concept is used.

In the first approach, the computation starts as if a standard problem is solved. First the inviscid solver is run, and the resulting edge velocity distribution is input to the boundary layer solver, from which a displacement thickness distribution is obtained, which is then used as a boundary condition for the inviscid solver. At the point where a separation is observed ( $\equiv \tau_{w} \leq 0$ ) , extrapolation for the displacement thickness is done. This procedure is repeated until convergence is achieved.

Major drawback of this method is that it can handle small separated zones, essentially separation bubbles, but is not good enough for long separated regions such as stalled airfoils, since the obtained results will deviate significantly from the correct ones.

The latter method is the employment of the blowing velocity concept which is referred to as the strong interaction problem and is used commonly by T. Cebeci et al $[3,4,12,13$, 14]. In this method, both the edge velocity distribution and the displacement thickness distribution are treated as unknowns. Equations are solved in inverse mode applying successive sweeps to the calculations. The idea is to alter the inviscid edge velocity distribution using Veldman's relation which is described thoroughly in reference [15].
$U_{e}(x)=U_{e}^{0}(x)+\delta U_{e}(x)$

In the above formulation, the first term in the RHS represents inviscid velocity distribution where the second represents the perturbation velocity distribution which is caused by viscous effects, and may be calculated using the Hilbert Integral which is given below:
$\delta U_{e}(x)=\frac{1}{\pi} \int_{x_{a}}^{x_{b}} \frac{d}{d \xi}\left(U_{e} \delta^{*}\right) \frac{d \xi}{(x-\xi)}$

In the above formulation the bounds of the integral are actually the start and end points of the interacting region. The term $x$ is the stream wise distance where $\xi$ stands as a dummy variable used for integration. Apparently, $U_{e}(x)$ is the edge velocity distribution and $\delta^{*}$ represents the displacement thickness. Solution procedure is explained in detail in reference [1]. The Hilbert Integral is based upon thin airfoil theory and the blowing velocity $V_{n}$ which is:

$$
\begin{equation*}
V_{n}=\frac{d}{d \xi}\left(U_{e} \delta^{*}\right) \tag{1.1}
\end{equation*}
$$

Blowing velocity distribution is used as a boundary condition for the potential flow solver in order to include the effects of boundary layer in the inviscid flow solutions. Since the blowing velocity concept employs both the displacement thickness and the edge velocity distribution, it is superior to weak interaction approach which takes into account only the displacement thickness distribution. Thus, the blowing velocity concept provides a more complete mathematical model.

In order to clarify the definition one shall explain the displacement thickness phenomenon in more detail. The displacement thickness is defined by the expression shown in equation 3.4.
$\delta^{*}=\int_{0}^{\infty}\left[1-\frac{u(x, y)}{U_{e}(x)}\right] d y$

Where in the above expression, $u(x, y)$ represents the velocity component tangent to the wall. The physical meaning of the above relation may be explained as described in [1]. That is if the flow was purely inviscid, the mass flow rate through the boundary layer would be $\rho U_{e} \delta$, where $\delta$ represents the actual boundary layer thickness, normal to the wall. However, the flow being viscous, clearly much lesser amount of mass is passing through the same control volume. The thickness if the inviscid profile which would allow the same amount of mass transfer along itself is shown below in figure 3.1 with dashed line. Distance of this dashed line from the wall is called the displacement thickness and denoted by $y=\delta^{*}$. It shall also be noted that area I is equivalent to area II. This is shown by equation 3.5
$\rho U_{e} \delta^{*}=\rho \int_{0}^{\delta}\left(U_{e}-u\right) d y$


Figure 3.1 Definition of Displacement Thickness

Changing the upper limit of the integral in equation 3.5 from $\delta$ to infinity has no significant effect since at the edge of the boundary layer, it can be told that:

$$
\begin{equation*}
u(x, y) \cong U_{e}(x) \tag{3.6}
\end{equation*}
$$

The reasoning of the use of strong interaction approach can be explained as it simplifies the process bringing a relation between displacement thickness and edge velocity distribution as shown above.

### 3.2 Solution of the Standard Problem

Since the boundary layer equations are parabolic, marching technique is applicable starting from a known solution at say $x=x_{0}$. As one proceeds in downstream direction, since the thickness of the boundary layer increases, one has to maintain sufficiently high resolution, in order to keep increments in stream-wise direction small, in order to maintain computational accuracy. In order to solve equation 2.19, one may solve it as it is, or introduce a more convenient form by using a similarity transformation which eliminates the problems related to the boundary layer growth phenomenon and therefore the restrictions in stream-wise grid spacing.

A widely used similarity transformation is the Falkner-Skan transformation, which is a consequence of group theoretic method described by Hansen in reference [16]. In Falkner-Skan transformation, $x$ values remain unchanged, even in some cases they are denoted by the Greek letter $\xi$. The similarity variables used in Falkner-Skan transformation is the dimensionless distance in normal direction that is y-direction, $\eta$ which is given below in equation 3.7.

$$
\begin{equation*}
\eta=\sqrt{\frac{U_{e}}{v x}} y \tag{3.7}
\end{equation*}
$$

And the corresponding dimensionless stream function $f(x, \eta)$ is given in equation 3.8 below.
$\psi(x, y)=\sqrt{U_{e} v x} f(x, \eta)$

In equation 3.8, $\psi$ is the dimensional stream function, from which the velocity components may be written as shown below in equation 3.9.
$u=\frac{\partial \psi}{\partial y}$
$\nu=-\frac{\partial \psi}{\partial x}$

Using the prescribed similarity transformation, differentiation with respect to x and y terms become:
$\left(\frac{\partial}{\partial x}\right)_{y}=\left(\frac{\partial}{\partial \eta}\right)_{x} \frac{\partial}{\partial x}+\left(\frac{\partial}{\partial x}\right)_{\eta}$
$\left(\frac{\partial}{\partial y}\right)_{x}=\left(\frac{\partial}{\partial \eta}\right)_{x} \frac{\partial}{\partial y}$

Falkner-Skan transformation enables one to generate initial condition at and near the stagnation point, and results in the boundary layer equations to become ordinary differential equations rather than partial ones for similar flows. However, its use is not limited to similar flows since even if it does not eliminate, it significantly decreases the dependence in stream-wise position. The latter is valid for the case investigated as the presence of x as a variable can be recognized from equation 3.10.

Defining $\xi$ as the dimensionless surface distance as $\xi=x / L, L$ being the characteristic length of the flow, and applying the above transformation to equation 2.19 yields equation 3.11 which is given below.
$\left(b f^{\prime \prime}\right)^{\prime}+\frac{m+1}{2} f f^{\prime \prime}+m\left[1-\left(f^{\prime}\right)^{2}\right]=\xi\left(f^{\prime} \frac{\partial f^{\prime}}{\partial \xi}-f^{\prime \prime} \frac{\partial f}{\partial \xi}\right)$

And the corresponding boundary conditions are given below in equation 3.12.

$$
\begin{array}{ll}
f^{\prime}=0, f=0 & \text { at } \eta=0 \\
f^{\prime}=1 & \text { at } \eta=\eta_{e} \tag{3.12b}
\end{array}
$$

In equation 3.12, $\eta_{e}$ is the dimensionless coordinate corresponding to the edge of the boundary layer as stated in [7]. The definition is given below in equation 3.13.
$\eta_{e}=\sqrt{\frac{R_{L} U_{e}}{\xi}} \frac{\delta}{L}$

In equation 3.11, primes denote differentiation with respect to the transformation variable $\eta$. Also one shall note that $m$ is the dimensionless pressure gradient defined below in equation 3.14 and $v_{t}^{+}$being the fraction of eddy viscosity to kinematic viscosity of the fluid. Notice that, definition for the term $b$ is altered to make it also dimensionless as defined by equation 3.16.
$m=\frac{\xi}{U_{e}} \frac{d U_{e}}{d \xi}$
$v_{t}^{+}=\frac{\varepsilon_{m}}{v}$
$b=1+v_{t}^{+}$

Equation 3.16 is valid both for laminar and turbulent flows, and its value becomes unity in case of laminar flows.

In order to solve equation 3.11 together with its boundary conditions, Keller's box scheme is employed as explained above. This scheme is essentially a two point finitedifference scheme, in which the equation itself and the boundary conditions are expressed as a system of three equations of the first-order. In order to achieve this goal, new variables are introduced to express derivatives of $f$. In Keller's box scheme, the variables are evaluated at the vertices, where the equations are written for the center of the box.


Figure 3.2 Rectangular grid used for centered difference approximation.

The nodes of the above rectangular grid are chosen as:

$$
\begin{array}{ll}
\xi_{n}=\xi_{n-1}+k_{n} & n=1,2,3, \cdots, N \\
\eta_{j}=\eta_{j-1}+h_{j} & j=1,2,3, \cdots, J
\end{array}
$$

As the resultant system of equations is non-linear, Newton's method, which is introduced in subsection 3.2.2 is used for linearization, and then solved by the block elimination method described in subsection 3.2.3.

### 3.2.1 Numerical Scheme

As described above in section 3.2 new variables are introduced to represent derivatives of $f$, enabling one to write equation 3.11 and its associated boundary conditions 3.12 as a system of first order equations. Note that the newly introduced variables are not velocity components but the derivatives of the non-dimensional stream function $f$ with respect to similarity variable $\eta$. Defining $u(x, \eta)$ and $v(x, \eta)$ such that:
$f^{\prime}=u$
$f^{\prime \prime}=u^{\prime}=v$

Using these new variables, eqs.3.11 and 3.12 are rewritten as eqs.3.19 and 3.20.

$$
\begin{array}{ll}
(b v)^{\prime}+\frac{m+1}{2} f v+m\left[1-u^{2}\right]=\xi\left(u \frac{\partial u}{\partial \xi}-v \frac{\partial f}{\partial \xi}\right) \\
u=0, \quad f=0 & \text { at } \eta=0 \\
u=1 & \text { at } \eta=\eta_{e}
\end{array}
$$

Writing equation 3.18 in discrete form using Keller's box method yields:

$$
\begin{align*}
& \frac{f_{j}^{n}-f_{j-1}^{n}}{h_{j}}=\frac{u_{j}^{n}+u_{j-1}^{n}}{2} \equiv u_{j-1 / 2}^{n}  \tag{3.21a}\\
& \frac{u_{j}^{n}-u_{j-1}^{n}}{h_{j}}=\frac{v_{j}^{n}+v_{j-1}^{n}}{2} \equiv v_{j-1 / 2}^{n} \tag{3.21b}
\end{align*}
$$

With $L$ denoting the LHS, and $R$ denoting the RHS, equation 3.19 can be approximated at the center of the rectangle by first centering in $\xi$ direction as shown below in equation 3.22.

$$
\begin{equation*}
\frac{1}{2}\left(L^{n}+L^{n-1}\right)=\xi^{n-1 / 2}\left[u^{n-1 / 2}\left(\frac{u^{n}-u^{n-1}}{k^{n}}\right)-v^{n-1 / 2}\left(\frac{f^{n}-f^{n-1}}{k^{n}}\right)\right] \tag{3.22}
\end{equation*}
$$

Where:

$$
\begin{align*}
& L^{n}=\left[(b v)^{\prime}+\frac{m+1}{2} f v+m\left[1-u^{2}\right]\right]^{n}  \tag{3.23a}\\
& L^{n-1}=\left[(b v)^{\prime}+\frac{m+1}{2} f v+m\left[1-u^{2}\right]\right]^{n-1} \tag{3.23b}
\end{align*}
$$

Defining $\alpha^{n}$ as in equation 3.24 one can rewrite equation 3.22 as shown in equation 3.25.
$\alpha^{n}=\frac{\xi^{n-1 / 2}}{k^{n}}$
$L^{n}-\alpha^{n}\left[\left(u^{2}\right)^{n}-v^{n} f^{n}-v^{n-1} f^{n}+v^{n} f^{n-1}\right]=-L^{n-1}+\alpha^{n}\left[-\left(u^{2}\right)^{n-1}+f^{n-1} v^{n-1}\right]$

Since the values having the superscript $\left({ }^{n-1}\right)$ are known from the previous station, and one is trying to obtain the values of current station, one may rearrange terms in equation 3.22 and obtain:
$\left[(b v)^{\prime}\right]^{n}+\alpha_{1}(f v)^{n}-\alpha_{2}\left(u^{2}\right)^{n}+\alpha^{n}\left[\left(f^{n} v^{n-1}\right)-v^{n} f^{n-1}\right]=R^{n-1}$

In the above equation the newly introduced variables are:
$\alpha_{1}=\frac{m^{n}+1}{2}+\alpha^{n}$
$\alpha_{2}=m^{n}+\alpha^{n}$
$R^{n-1}=-L^{n-1}+\alpha^{n}\left[v^{n-1} f^{n-1}-\left(u^{2}\right)^{n-1}\right]-m^{n}$

As now the equation 3.19 is centered in $\xi$ direction, one may proceed to center it in $\eta$ direction, that is to write it at $\eta_{j-1 / 2}$.
$\frac{\left(b_{j} v_{j}\right)^{n}-\left(b_{j-1} v_{j-1}\right)^{n}}{h_{j}}+\alpha_{1}(f v)_{j-1 / 2}^{n}-\alpha_{2}\left(u^{2}\right)_{j-1 / 2}^{n}+\alpha^{n}\left[f_{j-1 / 2}^{n} v_{j-1 / 2}^{n-1}-f_{j-1 / 2}^{n-1} v_{j-1 / 2}^{n}\right]=R_{j-1 / 2}^{n-1}$

Where:
$R_{j-1 / 2}^{n-1}=-L_{j-1 / 2}^{n-1}+\alpha^{n}\left[v_{j-1 / 2}^{n-1} f_{j-1 / 2}^{n-1}-\left(u^{2}\right)_{j-1 / 2}^{n-1}\right]-m^{n}$
$L_{j-1 / 2}^{n-1}=\left\{\frac{\left(b_{j} v_{j}\right)-\left(b_{j-1} v_{j-1}\right)}{h_{j}}+\frac{m+1}{2}(f v)_{j-1 / 2}+m\left[1-\left(u^{2}\right)_{j-1 / 2}\right]\right\}^{n-1}$

Note that equations 3.21 and 3.28 are imposed for $j=1,2,3, \cdots, J-1$ at any given $\eta$. For $j=0$ and $j=J$, the boundary conditions given in equation 3.20 are used. The transformed boundary layer thickness, $\eta_{e}$, has to be sufficiently large, so that $U_{e} \rightarrow 1$ asymptotically. This is usually satisfied when $v\left(\eta_{e}\right)<10^{-3}$.

Also the boundary conditions given in equation 3.20 can be written in discrete form as shown in equation 3.30.

$$
\begin{array}{ll}
f_{0}^{n}=0 & u_{0}^{n}=0 \\
u_{J}^{n}=1 & \text { at } j=0  \tag{3.30b}\\
\text { at } j=J
\end{array}
$$

### 3.2.2 Newton's Method

As the above derived discretized system of equations is non-linear, one has to linearize them in order to solve them in matrix form. In order to accomplish this objective, small perturbation quantities for all of the variables are introduced, and an iterative scheme called Newton's method is employed.

As an initial guess, values of the variables at the previous station are used. Note that in the below expression, $k$ stands for the order of iteration, $\delta$ represents perturbation quantities.
$f_{j}^{k+1}=f_{j}^{k}+\delta f_{j}^{k}$
$u_{j}^{k+1}=u_{j}^{k}+\delta u_{j}^{k}$
$v_{j}^{k+1}=v_{j}^{k}+\delta v_{j}^{k}$

After substituting the above quantities into eqns.3.21 and 3.28, one obtains the following set of equations after neglecting higher order terms.

$$
\begin{align*}
& \delta f_{j}^{n}-\delta f_{j-1}^{n}-\frac{h_{j}}{2}\left(\delta u_{j}^{n}+\delta u_{j-1}^{n}\right)=\frac{h_{j}}{2}\left(u_{j}^{n}-u_{j-1}^{n}\right)-\left(f_{j}^{n}-f_{j-1}^{n}\right)=\left(r_{1}\right)_{j}  \tag{3.32}\\
& \delta u_{j}^{n}-\delta u_{j-1}^{n}-\frac{h_{j}}{2}\left(\delta v_{j}^{n}+\delta v_{j-1}^{n}\right)=\frac{h_{j}}{2}\left(v_{j}^{n}-v_{j-1}^{n}\right)-\left(u_{j}^{n}-u_{j-1}^{n}\right)=\left(r_{3}\right)_{j-1} \tag{3.33}
\end{align*}
$$

$\left(s_{1}\right)_{j} \delta v_{j}^{n}+\left(s_{2}\right)_{j} \delta v_{j-1}^{n}+\left(s_{3}\right)_{j} \delta f_{j}^{n}+\left(s_{4}\right)_{j} \delta f_{j-1}^{n}+\left(s_{5}\right)_{j} \delta u_{j}^{n}+\left(s_{6}\right)_{j} \delta u_{j-1}^{n}=\left(r_{2}\right)_{j}$

Where $r$ terms are:
$\left(r_{1}\right)_{j}=\left(f_{j-1}-f_{j}\right)+h_{j} u_{j-1 / 2}$
$\left(r_{3}\right)_{j}=\left(u_{j-1}-u_{j}\right)+h_{j} v_{j-1 / 2}$
$\left(r_{2}\right)=R_{j-1 / 2}^{n-1}-\frac{\left(b_{j} v_{j}-b_{j-1} v_{j-1}\right)^{n}}{h_{j}}-\alpha_{1}(f v)_{j-1 / 2}^{n}+\alpha_{2}\left(u^{2}\right)_{j-1 / 2}^{n}-\alpha^{n} v_{j-1 / 2}^{n-1} f_{j-1 / 2}^{n}+\alpha^{n} f_{j-1 / 2}^{n-1} v_{j-1 / 2}^{n}$

The coefficients of the perturbation quantities appearing above in equation 3.34 are as given in equation 3.36.

$$
\begin{equation*}
\left(s_{1}\right)_{j}=h_{j}^{-1} b_{j}^{n}+\frac{\alpha_{1}}{2} f_{j}^{n}-\frac{\alpha^{n}}{2} f_{j-1 / 2}^{n-1} \tag{3.36a}
\end{equation*}
$$

$\left(s_{2}\right)_{j}=-h_{j-1}^{-1} b_{j-1}^{n}+\frac{\alpha_{1}}{2} f_{j-1}^{n}-\frac{\alpha^{n}}{2} f_{j-1 / 2}^{n-1}$
$\left(s_{3}\right)_{j}=\frac{\alpha_{1}}{2} v_{j}^{n}+\frac{\alpha^{n}}{2} v_{j-1 / 2}^{n-1}$
$\left(s_{4}\right)_{j}=\frac{\alpha_{1}}{2} v_{j-1}^{n}+\frac{\alpha^{n}}{2} v_{j-1 / 2}^{n-1}$

$$
\begin{equation*}
\left(s_{5}\right)_{j}=-\alpha_{2} u_{j}^{n} \tag{3.36e}
\end{equation*}
$$

$$
\begin{equation*}
\left(s_{6}\right)_{j}=-\alpha_{2} u_{j-1}^{n} \tag{3.36f}
\end{equation*}
$$

The boundary conditions also change as the perturbation equations applied, and become as shown below:
$f_{0}^{n}+\delta f_{0}^{n}=f_{w} \quad \delta f_{0}^{n}=0$
$u_{0}^{n}+\delta u_{0}^{n}=0$

$$
\begin{equation*}
\delta u_{0}^{n}=0 \tag{3.37b}
\end{equation*}
$$

$u_{J}^{n}+\delta u_{J}^{n}=1$

$$
\begin{equation*}
\delta u_{J}^{n}=0 \tag{3.37c}
\end{equation*}
$$

Equations 3.32-3.34 are written for each j-station starting from the wall, extending to the edge of the boundary layer and will be represented in matrix notation as in equation 3.38.
$[\underset{\sim}{A}] \cdot\{\underset{\sim}{\delta}\}=\{\underset{\sim}{r}\}$

Defining terms in the above equation:
$A$ is the tri-diagonal coefficient matrix defined as:
$[\underset{\sim}{A}]=\left[\begin{array}{cccccccc}A_{0} & C_{o} & & & & & & \\ B_{1} & A_{1} & C_{1} & & & & & \\ & B_{2} & A_{2} & C_{2} & & & & \\ & & \ddots & \ddots & \ddots & & & \\ & & & B_{j} & A_{j} & C_{j} & & \\ & & & & \ddots & \ddots & \ddots & \\ & & & & & B_{J-1} & A_{J-1} & C_{J-1} \\ & & & & & & B_{J} & A_{J}\end{array}\right]$

And $\delta$ is the vector of unknowns.

$$
[\delta]=\left\{\begin{array}{c}
\delta_{0}  \tag{3.40a,b}\\
\delta_{1} \\
\vdots \\
\delta_{j} \\
\vdots \\
\delta_{J}
\end{array}\right\} \quad[\underset{\sim}{r}]=\left\{\begin{array}{c}
r_{0} \\
r_{1} \\
\vdots \\
r_{j} \\
\vdots \\
r_{J}
\end{array}\right\}
$$

The elements in equation 3.40 are vectors defined as follows:

$$
\delta_{j}=\left\{\begin{array}{l}
\delta f_{j}  \tag{3.41a,b}\\
\delta u_{u} \\
\delta v_{j}
\end{array}\right\} \quad r_{j}=\left\{\begin{array}{l}
\left(r_{1}\right)_{j} \\
\left(r_{2}\right)_{j} \\
\left(r_{3}\right)_{j}
\end{array}\right\} \quad 0 \leq j \leq J
$$

In addition, elements of matrix A shown in equation 3.39 are sub-matrices for which definitions are given below in equation 3.42.

$$
\begin{align*}
& A_{0}=\left[\begin{array}{ccc}
1 & 0 & 0 \\
0 & 1 & 0 \\
0 & -1 & -h_{1} / 2
\end{array}\right]  \tag{3.42a}\\
& A_{j}=\left[\begin{array}{ccc}
-1 & -h_{j} / 2 & 0 \\
\left(s_{3}\right)_{j} & \left(s_{5}\right)_{j} & \left(s_{1}\right)_{j} \\
0 & -1 & -h_{j+1} / 2
\end{array}\right]  \tag{3.42b}\\
& A_{J}=\left[\begin{array}{ccc}
1 & -h_{J} / 2 & 0 \\
\left(s_{3}\right)_{J} & \left(s_{5}\right)_{J} & \left(s_{1}\right)_{J} \\
0 & 1 & 0
\end{array}\right]  \tag{3.42c}\\
& B_{j}=\left[\begin{array}{ccc}
-1 & -h_{j} / 2 & 0 \\
\left(s_{4}\right)_{j} & \left(s_{6}\right)_{j} & \left(s_{2}\right)_{j} \\
0 & 0 & 0
\end{array}\right] \tag{3.42d}
\end{align*}
$$

$$
C_{j}=\left[\begin{array}{ccc}
0 & 0 & 0  \tag{3.42e}\\
0 & 0 & 0 \\
0 & 1 & -h_{j+1} / 2
\end{array}\right] \quad 0 \leq j \leq J-1
$$

### 3.3 Inverse Problem

Since the standard problem can not handle separated regions as in attached regions, the edge velocity can not be calculated, one has to switch from the standard problem to the inverse problem. In this study, this switching is automatically done by the computer code utilized after a few stations in x-direction. For further information see reference [1].

The procedure described above in section 3.2 will be modified to handle the inverse problem. When the displacement thickness is specified, one can establish a relationship between the displacement thickness and the edge velocity. In order to establish such a relationship the solution procedure is modified as described in subsection 3.3.1.
3.3.1 Solution Procedure for Specified Displacement Thickness

Consider an external flow case where the displacement thickness distribution is given as $\delta^{*}(x)$. The boundary conditions for such a problem would be:
$y=0$

$$
\begin{equation*}
u=0, v=0 \tag{3.43a}
\end{equation*}
$$

$y=\delta ;$

$$
\begin{equation*}
u=u_{e}, \delta^{*}(x) \text { as specified. } \tag{3.43b}
\end{equation*}
$$

One shall follow the same procedure described in section 3.2 and start with a transformation. The difference is that, in this case, since the edge velocity distribution $U_{e}$, is unknown and therefore can not be used in the definition of the transformation variable. For this reason, the edge velocity distribution is replaced by the free-stream velocity, $U_{\infty}$, so that the new transformation becomes:
$Y=\sqrt{\frac{U_{\infty}}{v x}} y$
$\psi=\sqrt{U_{\infty} v x} F(\xi, Y)$

Using these new transformation variables, the boundary layer equations and the corresponding boundary conditions are rewritten as below.
$\left(b F^{\prime \prime}\right)^{\prime}+\frac{1}{2} F F^{\prime \prime}=\xi\left(F^{\prime} \frac{\partial F^{\prime}}{\partial \xi}-F^{\prime \prime} \frac{\partial F}{\partial \xi}\right)-\xi \overline{U_{e}} \frac{d \overline{U_{e}}}{d \xi}$
$Y=0$
$F^{\prime}=0, F=0$
$Y=Y_{e}$
$F_{e}^{\prime}=\overline{U_{e}}(\xi), \frac{F_{e}}{F_{e}^{\prime}}=Y_{e}-\frac{\delta^{*}(\xi)}{L} \frac{\sqrt{R_{L}}}{\sqrt{\xi}} \equiv e(\xi)$

Where in the above formulations, L is the characteristic length and primes denote differentiation with respect to transformation variable, Y. Other newly introduced variables are given in equation 3.47.
$\xi=\frac{x}{L}$,
$R_{L}=\frac{U_{\infty} L}{v}$,
$\overline{U_{e}}=\frac{U_{e}}{U_{\infty}}$

For the edge boundary condition, we may use the definition of displacement thickness as shown in equation 3.48 if its value is known prior to these calculations.
$\delta^{*}(\xi)=\frac{\sqrt{\xi} L}{\sqrt{R_{L}}}\left(Y_{e}-\frac{F_{e}}{F_{e}^{\prime}}\right)$

Since in the inverse problem value of $\overline{U_{e}}$ is unknown, it has to be integrated into the solution. In order to do it, a new variable is introduced.
$w=U_{e}(\xi)$

And since the edge velocity is a function of $\xi$ only: $w(\xi)$, one may write the following equation.
$w^{\prime}=0$

The next step is to reduce the order of the partial differential equation (PDE) of equation 3.45 as in equation 3.51.
$F^{\prime}=U$
$F^{\prime \prime}=U^{\prime}=V$
$(b V)^{\prime}+\frac{1}{2} F V=\xi\left(U \frac{\partial U}{\partial \xi}-V \frac{\partial F}{\partial \xi}\right)-\xi w \frac{d w}{d \xi}$
$w^{\prime}=0$

According to the new PDE, the new boundary conditions are:
$Y=0$
$F=U=0$
$Y=Y_{e}$
$U=w, \quad F=e(\xi) w$

In order to solve these equations in a rectangular grid, one has to discretize them by employing finite difference approximations.
$\frac{F_{j}-F_{j-1}}{h_{j}}=U_{j-1 / 2}$
$\frac{U_{j}-U_{j-1}}{h_{j}}=V_{j-1 / 2}$

$$
\begin{align*}
& \frac{w_{j}^{n}-w_{j-1}^{n}}{h_{j}}=0  \tag{3.53c}\\
& h_{j}^{-1}\left(b_{j}^{n} V_{j}^{n}-b_{j-1}^{n} V_{j-1}^{n}\right)+\left(\frac{1}{2}+\alpha^{n}\right)(F V)_{j-1 / 2}^{n}+\alpha^{n}\left(V_{j-1 / 2}^{n-1} F_{j-1 / 2}^{n}-F_{j-1 / 2}^{n-1} V_{j-1 / 2}^{n}\right)=R_{j-1 / 2}^{n-1} \tag{3.53d}
\end{align*}
$$

Where

$$
\begin{align*}
& R_{j-1 / 2}^{n-1}=-L_{j-1 / 2}^{n}+\alpha^{n}\left[(F V)_{j-1 / 2}^{n}-\left(u^{2}\right)_{j-1 / 2}^{n-1}\right]  \tag{3.54a}\\
& L_{j-1 / 2}^{n-1}=\left\{\frac{\left(b_{j} V_{j}-b_{j-1} V_{j-1}\right)}{h_{j}}+\frac{1}{2}(F V)_{j-1 / 2}-\alpha^{n}\left(w^{2}\right)_{j-1 / 2}\right\}^{n-1} \tag{3.54b}
\end{align*}
$$

Note that in equation 3.54b, $n-1$ in the RHS means, value at station $n-1$.

If one compares these equations with those of equation 3.26 , it is clear that of one sets $m=0$ in eqs.3.27, the results will be:

$$
\begin{equation*}
\alpha_{1}=\frac{1}{2}+\alpha^{n} \tag{3.55a}
\end{equation*}
$$

$$
\begin{equation*}
\alpha_{2}=\alpha^{n} \tag{3.55b}
\end{equation*}
$$

The final step is to linearize these equations in the same manner as in subsection 3.2.2. Note that there is one new variable $w$, and hence a corresponding perturbation has to be introduced for $w$.

$$
\begin{align*}
& \delta F_{j}^{n}-\delta F_{j-1}^{n}-\frac{h_{j}}{2}\left(\delta U_{j}^{n}+\delta U_{j-1}^{n}\right)=\frac{h_{j}}{2}\left(U_{j}^{n}-U_{j-1}^{n}\right)-\left(F_{j}^{n}-F_{j-1}^{n}\right)=\left(r_{1}\right)_{j}  \tag{3.56a}\\
& \delta U_{j}^{n}-\delta U_{j-1}^{n}-\frac{h_{j}}{2}\left(\delta V_{j}^{n}+\delta V_{j-1}^{n}\right)=\frac{h_{j}}{2}\left(V_{j}^{n}-V_{j-1}^{n}\right)-\left(U_{j}^{n}-U_{j-1}^{n}\right)=\left(r_{3}\right)_{j-1}  \tag{3.56b}\\
& \delta w_{j}-\delta w_{j-1}=\left(w_{j-1}-w_{j}\right) \tag{3.56c}
\end{align*}
$$

Recall that $r$ terms are:

$$
\begin{align*}
& \left(r_{1}\right)_{j}=\left(F_{j-1}-F_{j}\right)+h_{j} U_{j-1 / 2}  \tag{3.35a}\\
& \left(r_{3}\right)_{j}=\left(U_{j-1}-U_{j}\right)+h_{j} V_{j-1 / 2}  \tag{3.35b}\\
& \left(r_{2}\right)=R_{j-1 / 2}^{n-1}-\frac{\left(b_{j} V_{j}-b_{j-1} V_{j-1}\right)^{n}}{h_{j}}-\alpha_{1}(F V)_{j-1 / 2}^{n}+\alpha_{2}\left(U^{2}\right)_{j-1 / 2}^{n}-\alpha^{n} V_{j-1 / 2}^{n-1} F_{j-1 / 2}^{n}+\alpha^{n} F_{j-1 / 2}^{n-1} V_{j-1 / 2}^{n} \tag{3.35c}
\end{align*}
$$

Equation 3.57 is written very much like equation 3.34 except for two additional terms coming from the introduction of our new variable.
$\left(s_{1}\right)_{j} \delta V_{j}+\left(s_{2}\right)_{j} \delta V_{j-1}+\left(s_{3}\right)_{j} \delta F_{j}+\left(s_{4}\right)_{j} \delta F_{j-1}+\left(s_{5}\right)_{j} \delta U_{j}$
$+\left(s_{6}\right)_{j} \delta U_{j-1}+\left(s_{7}\right)_{j} \delta w_{j}+\left(s_{8}\right)_{j} \delta w_{j-1}=\left(r_{2}\right)_{j}$

And the definitions of the $s$ terms are given by equation 3.36 , except for $\left(s_{7}\right)_{j}$ and $\left(s_{8}\right)_{j}$, which are given by equation 3.58 .
$\left(s_{1}\right)_{j}=h_{j}^{-1} b_{j}^{n}+\frac{\alpha_{1}}{2} f_{j}^{n}-\frac{\alpha^{n}}{2} f_{j-1 / 2}^{n-1}$
$\left(s_{2}\right)_{j}=-h_{j-1}^{-1} b_{j-1}^{n}+\frac{\alpha_{1}}{2} f_{j-1}^{n}-\frac{\alpha^{n}}{2} f_{j-1 / 2}^{n-1}$
$\left(s_{3}\right)_{j}=\frac{\alpha_{1}}{2} v_{j}^{n}+\frac{\alpha^{n}}{2} v_{j-1 / 2}^{n-1}$
$\left(s_{4}\right)_{j}=\frac{\alpha_{1}}{2} v_{j-1}^{n}+\frac{\alpha^{n}}{2} v_{j-1 / 2}^{n-1}$
$\left(s_{5}\right)_{j}=-\alpha_{2} u_{j}^{n}$
$\left(s_{6}\right)_{j}=-\alpha_{2} u_{j-1}^{n}$

$$
\begin{equation*}
\left(s_{7}\right)_{j}=\alpha^{n} w_{j} \tag{3.58a}
\end{equation*}
$$

$\left(s_{8}\right)_{j}=\alpha^{n} w_{j-1}$

The new, linearized boundary conditions are as of equation 3.59.
$\delta F_{0}=0$
$\delta U_{0}=0$
$\delta U_{J}-\delta w_{J}=w_{J}-U_{J}$
$\delta F_{J}-e^{n} \delta w_{J}=e^{n}\left(w_{J}\right)-F_{J}$

The above discrete equation set is written in matrix form just as in section 3.2. except that sub-matrices in $[\underset{\sim}{A}]$ will be $4 \times 4$ rather that $3 \times 3$.

$$
\begin{equation*}
[\underset{\sim}{A}] \cdot\{\underset{\sim}{\delta}\}=\{\underset{\sim}{r}\} \tag{3.38}
\end{equation*}
$$

$[\underset{\sim}{A}]$ is a tri-diagonal matrix with the same form of equation 3.39, except that sub-matrices in $[\underset{\sim}{A}]$ will be $4 \times 4$ rather that $3 \times 3$. Fist two rows of the element matrices $A_{0}$ and $C_{0}$, and the last two rows of the element matrices $B_{J}$ and $A_{J}$ constitute the boundary conditions. Likewise, the vectors $\{\underset{\sim}{\delta}\}$ and $\{\underset{\sim}{r}\}$ remain the same with element vectors of them $\delta_{j}$ and $r_{j}$ are now $4 \times 1$ vectors.

The solution method is again the block elimination method, which is explained in appendix A.

### 3.3.2 Interaction Procedure

In order to solve separated flows with the inverse method described above, one has to specify either the displacement thickness or the wall shear as noted in [9]. In order to
solve for external flows as intended in this study, a potential flow solver is required to calculate the edge velocity distribution, which enables one to solve the boundary layer equations in inverse mode. The interaction is accomplished via the blowing velocity distribution as described in equation 1.1. The blowing velocity itself, serves as a boundary condition for the next step of the potential flow calculations.

In order to solve the boundary layer equations, the blowing velocity concept in Veldman's suggestion is used as discussed in section 3.1. Rewriting equation 3.2 using definition given in equation 3.3, one obtains equation 3.60.
$U_{e}(x)=U_{e}^{0}(x)+\delta U_{e}(x)$
$U_{e}\left(x_{i}\right)=U_{e}^{0}\left(x_{i}\right)+\frac{1}{\pi} \int \frac{d}{d \xi}\left(U_{e} \delta^{*}\right) \frac{d \xi}{\left(x_{i}-\xi\right)}$

The Hilbert integral in the above equation can be approximated as described in appendix $B$, to yield:
$U_{e}\left(x_{i}\right)=U_{e}^{0}\left(x_{i}\right)+\sum_{j=1}^{N} C_{i j}\left(x_{i}\right) \delta^{*}\left(x_{j}\right)$
$U_{e}\left(x_{i}\right)-C_{i i} \delta^{*}\left(x_{i}\right)=U_{e}^{0}\left(x_{i}\right)+\sum_{j \neq i} C_{i j} \delta^{*}\left(x_{j}\right)=g_{i}$

Rewriting equation 3.48 as in equation 3.63 and substituting into equation 3.62 , one may get equation 3.64.

$$
\begin{align*}
& \delta^{*}\left(x_{i}\right)=y_{e}-\frac{\psi_{e}}{U_{e}}=\frac{\sqrt{\xi}}{\sqrt{R_{L}}}\left(Y_{e}-\frac{F_{e}}{U_{e}}\right)  \tag{3.63}\\
& U_{e}\left(x_{i}\right)-C_{i i}\left[y_{e}-\frac{\psi_{e}}{U_{e}}\right]=g_{i} \tag{3.64}
\end{align*}
$$

Since the above formulation is non-linear, one has to linearize it in $\psi_{e}$ and $U_{e}$ to use as a boundary condition.
$U_{e}+\delta U_{e}-C_{i i}[p+\delta p]=g_{i}$

Where the term $p$ and $\delta p$ in the above equation is defined by equation 3.66
$p=y_{e}-\frac{\psi_{e}\left(x_{i}\right)}{U_{e}\left(x_{i}\right)}$
$\delta p=\frac{-\delta \psi_{e}}{U_{e}}+\frac{\psi_{e}}{U_{e}^{2}} \delta U_{e}$

Hence the linearized equation becomes equation 3.68.

$$
\begin{equation*}
\left(\frac{C_{i i}}{U_{e}}\right) \delta \psi_{e}+\left(1+C_{i i} \frac{\psi_{e}}{U_{e}^{2}}\right) \delta U_{e}=g_{i}-U_{e}+C_{i i}\left(y_{e}-\frac{\psi_{e}}{U_{e}}\right) \tag{3.68}
\end{equation*}
$$

From the above equations, one can easily note that in order to calculate $g_{i}, C_{i j}$ and $\delta^{*}\left(x_{j}\right)$ have to be known. In order to calculate them, sweeps within the boundary layer are performed. For $j>i$, the displacement thickness values are taken from the previous sweep and in case $j<i$, values calculated in the current sweep are used. With this additional information, one can now proceed to calculate the summation term in equation 3.62 in an iterative manner. Once it is converged, the calculations proceed to the next x station where the whole process is repeated. For calculation of coefficients $C_{i i}$, refer to appendix B .

### 3.4 Solution Procedure for the Wake Region

As described thoroughly in [1], solution procedure for the wake region is much more complex compared to solution on the body itself. In order to eliminate problems related to transition from no-slip wall boundary condition to smooth flow in the wake region, one has to prepare an extremely fine grid near the trailing edge of a lifting section in order
to impose this change gradually to the solution domain. One of the major difficulties in wake calculation is that since at high angles-of-attack, the boundary layer coming from the upper surface is turbulent, separated and thick compared to a thin, laminar or transitional boundary layer coming from the lower surface, merging process of these boundary layers is complicated, and may be considered like a mixing layer with considerable backflow.

In order to solve the boundary layer equations within the wake region, one has to recall equation 2.19.
$u \frac{\partial u}{\partial x}+v \frac{\partial u}{\partial y}=U_{e} \frac{d U_{e}}{d x}+\frac{\partial}{\partial y}\left(b \frac{\partial u}{\partial y}\right)$

As in the case of wall boundary layers, the above equation is written as a system of first order equations as shown in equation 3.51.
$F^{\prime}=U$
$U^{\prime}=V$
$(b V)^{\prime}+\frac{1}{2} F V=\xi\left(U \frac{\partial U}{\partial \xi}-V \frac{\partial F}{\partial \xi}\right)-\xi w \frac{d w}{d \xi}$
$w^{\prime}=0$

However, the boundary conditions for the wake region are different, and are given as:

| $F^{\prime}=w$ | at | $Y=Y_{-e}$ |
| :--- | :--- | :--- |
| $F=0$ | at | $Y=0$ |
| $F^{\prime}=w$ | at | $Y=Y_{e}$ |
| $w-\tilde{C}_{i i}\left[w\left(Y_{e}-Y_{-e}\right)-\left(F_{e}-F_{-e}\right)\right]=g_{i}$ |  |  |

Where

$$
\begin{equation*}
\tilde{C}_{i i}=C_{i i}\left(\frac{v \xi}{U_{0}}\right)^{\frac{1}{2}} \tag{3.70}
\end{equation*}
$$

Where $Y_{e}$ and $Y_{-e}$ are the values of the transformation variable at the upper and lower edges of the boundary layer for the wake region respectively. Similarly, $F_{e}$ and $F_{-e}$ are the values for the dimensionless stream function for the upper and lower edges. In order to stabilize the solutions of the above system, its sensitivity to the boundary conditions involving the dimensionless stream function is reduced by employing the so called Mechul function prescribed in reference [3]. With the introduction of Mechul function, $F_{-e}$ is denoted by $s$. And as $s$ is a function of x only one may write,
$s^{\prime}=0$

The boundary conditions for the system given in equation 3.69 can now be rewritten as:
$F^{\prime}=w, s=F_{-e}$
at $\quad Y=Y_{-e}$
$F=0$
at

$$
\begin{equation*}
Y=0 \tag{3.72b}
\end{equation*}
$$

$\left.\begin{array}{l}F^{\prime}=w \\ w-\tilde{C}_{i i}\left[w\left(Y_{e}-Y_{-e}\right)-\left(F_{e}-s\right)\right]=g_{i}\end{array}\right\} \quad$ at $\quad Y=Y_{e}$

The solution procedure is pretty much the same as of the above sections, expect that the elements of matrix $[\underset{\sim}{A}]$, which are sub-matrices are now $5 \times 5$. Accordingly the vectors $\{\underset{\sim}{\delta}\}$ and $\{\underset{\sim}{r}\}$ become $5 \times 1$ vectors, and solved using the block elimination method described in appendix $A$.

## CHAPTER 4

## TRANSITION METHOD IN 2D INCOMPRESSIBLE FLOWS

Early attempts to understand turbulence phenomenon focused on the original laminar flow, and tried to explain the reasons of its end, which is the start of the turbulent flow.

Rayleigh introduced the inflectional instability and worked mainly on inviscid flows until Taylor and Prandtl introduced the effects of viscosity. A complete boundary layer instability theory was first introduced by Tollmien and Schlichting, who also calculate the amplification rates for most of the unstable frequencies. After the experiments carried out by Schubauer and Skramstad in 1947, which demonstrated the presence of two dimensional sinusoidal instability waves within a boundary layer, their connection with transition and quantitative description were given by the theory of Tollmien and Schlichting. It was in 1956 that the $\mathrm{e}^{\mathrm{n}}$ transition prediction method was introduced by Smith, Gamberon and van Ingen, which is still in use today and is the basis of this study. Pretch provided a large amount of numerical results by calculating the stability characteristics of Falkner-Skan velocity profiles in 1942. Further reading is available in reference [17].

The balance between stabilizing viscous forces and destabilizing shear forces are represented by the critical Reynolds number. Perturbations that lead to instabilities arise from small changes in the boundary conditions which may be due to free-stream turbulence, surface roughness, noise, etc. which constitute the so called disturbance environment. The problem about how these disturbances are entrained in the flow is related to the subject of receptivity.

Receptivity may be defined as the mechanisms that cause the above described disturbances to enter the flow which yield the creation of the initial amplitudes for the waves generating instabilities. These disturbances are usually very small so that they can not be measured using the modern instruments until the instabilities are developed. To date, this mechanism still remains as not fully understood.

The initial growth of the two dimensional Tollmien-Schlichting waves can be described by the linear stability theory which constitutes the basis of the famous Smith-van Ingen $e^{n}$ transition prediction method. After their growth, three dimensional waves and/or nonlinear interactions are observed, which means that normally the linear stability theory can no longer be used. However, a bracketing assumption that extends the use of linear stability theory until transition is employed. The transition data obtained from such an approach has been shown to agree acceptably for most of the engineering problems of interest.

Currently the most popular method for predicting transition which is accepted as an engineering tool is the so called Smith-van Ingen $e^{\mathrm{n}}$ transition prediction method. As briefly discussed in chapter 1, it is based on solution of two-dimensional linear stability equations which are given by the fourth order ordinary differential equation (ODE) called the Orr-Sommerfeld equation given in equation 1.2. Derivation of this equation is explained in detail in Appendix C.

$$
\begin{equation*}
\left(\phi^{i v}-2 \alpha^{2} \phi^{\prime \prime}+\alpha^{4} \phi\right)=i R\left[(\alpha u-\omega)\left(\phi^{\prime \prime}-\alpha^{2} \phi\right)-\alpha u^{\prime \prime} \phi\right] \tag{1.2}
\end{equation*}
$$

In the above equation, primes denote differentiation in the normal direction, $u$ denotes the velocity profile in the stream-wise direction. Finally $\phi(y) \equiv\left(\phi_{r}+i \phi_{i}\right)$ represents the complex amplitude of the disturbance stream function $\hat{\psi}$ defined by the fluctuating velocity components $\hat{u}$ and $\hat{v}$ defined as shown in equation 4.1. In the below equation, hats stand for fluctuating quantities.
$\hat{u}=\frac{\partial \hat{\psi}}{\partial y}$

$$
\begin{equation*}
\hat{v}=-\frac{\partial \hat{\psi}}{\partial x} \tag{4.1}
\end{equation*}
$$

The parameter $\alpha$ is the wave-number of the disturbance which is related to the wavelength $\lambda$ via equation 4.2 , where $\omega$ is the circular frequency, whose unit is either radians per second or Hertz.
$\lambda=\frac{2 \pi}{\alpha}$

The case where the wave-number is complex $\left(\alpha \equiv \alpha_{r}+i \alpha_{i}\right)$ but the circular frequency is real is called the spatial amplification theory in which the amplitude of the disturbance varies with stream-wise position as $\exp \left(-\alpha_{i} x\right)$. On the other hand the case when the circular frequency is complex $\left(\omega \equiv \omega_{r}+i \omega_{i}\right)$, but the wave-number is real is called the temporal amplification theory in which the amplitude varies with time as $\exp \left(-\omega_{i} t\right)$. These two may be related to each other using the Gaster's transformation. These theories are discussed in great detail in references [5, 11, 17 and 18]. Also some more information is available in references $[19,20]$.

For the sake of convenience, quantities are transformed into their respective dimensionless forms. In order to non-dimensionalize the Orr-Sommerfeld equation given above, a dimensionless time variable is introduced as:
$\tau=t \frac{U_{\infty}}{L}$

Where $L$ is the reference length, defined as the characteristic length of the flow in the above sections, namely chord for external flows. Dividing all the velocity terms by the reference velocity, which is the free-stream velocity for external flows, and all lengths by the reference length, the dimensionless Orr-Sommerfeld equation becomes:
$\phi^{i v}-2 \alpha_{L}^{2} \phi^{\prime \prime}+\alpha_{L}^{4} \phi=i R\left[\left(\alpha_{L} \bar{u}-\bar{\omega}\right)\left(\phi^{\prime \prime}-\alpha_{L}^{2} \phi\right)-\alpha_{L} \bar{u}^{\prime \prime} \phi\right]$

Note that in the above equation primes denote differentiation with respect to dimensionless distance in the normal direction as $\frac{\partial}{\partial \bar{y}}$ where $y$ bar term is defined as $\bar{y}=\frac{y}{L}$. With the proper choice of $L$, the non-dimensional distance in the normal direction becomes the same as the one used in the boundary layer calculations. Also $\phi(\bar{y})$ is the complex amplitude of the disturbance stream function $\psi^{\prime}(\bar{x}, \bar{y}, \tau)$ defined as shown in equation 4.5 .
$\psi^{\prime}(\bar{x}, \bar{y}, \tau)=\phi(\bar{y}) \exp \left[i\left(\alpha_{L} \bar{x}-\bar{\omega} \tau\right)\right]$

Where
$\alpha_{L}=\alpha L$
$\bar{u}=\frac{u}{U_{\infty}}$
$\bar{\omega}=\frac{U_{\infty}}{L} \omega$
$R=\frac{U_{\infty} L}{v}$

In order to solve for equation 4.4, the boundary conditions are given in equation 4.7.

Wall B.C.: $\quad y=0 \quad \bar{y}=0, \phi=0, \phi^{\prime}=0$

Infinity B.C.: $\quad y \rightarrow \infty \quad \hat{u}=\hat{v}=0$

It is convenient to write the infinity boundary condition by taking into account that perturbation velocities will decay as the edge of the boundary layer is approached. In this case, the infinity boundary condition may be satisfied at the edge of the boundary layer by equation 4.8 .

$$
\left.\begin{array}{l}
\left(D^{2}-\varepsilon_{1}\right) \phi+\left(\varepsilon_{1}+\varepsilon_{2}\right)\left(D+\varepsilon_{1}\right) \phi=0 \\
\left(D+\varepsilon_{2}\right)\left(D^{2}-\varepsilon_{1}^{2}\right) \phi=0
\end{array}\right\} \quad \text { at } \quad \bar{y}=\frac{\delta}{L}
$$

Where
$D=\frac{d}{d \bar{y}}$
$\varepsilon_{1}=\alpha_{L}^{2}$
$\varepsilon_{2}^{2}=\varepsilon_{1}^{2}+i R\left(\alpha_{L} \bar{u}-\bar{\omega}\right)$

The onset of transition is obtained by solving the dimensionless Orr-Sommerfeld equation with give boundary conditions and following the $e^{n}$ procedure. The solution procedure may either be based on the spatial amplification theory or the temporal amplification theory.

In this study, the former one is preferred since the amplification of a disturbance can be measured in a steady mean flow. Since in the spatial amplification theory, the amplitude at a fixed point is independent of time, calculations of group velocities are not required. Also the spatial amplification theory gives the change in amplification values in a more direct manner than does the temporal theory as noted in [11].

### 4.1 Smith-van Ingen $e^{n}$ Procedure

The $e^{n}$ procedure requires the calculation of the amplification factors denoted by $\left(-\alpha_{i}\right)$ as a function of either the stream-wise position or Reynolds number based on the streamwise distance for a range of dimensional frequencies defined by equation 4.10.
$\omega^{*}=\bar{\omega} \frac{U_{\infty}}{L}$

The laminar boundary layer equations for a given external velocity distribution denoted by $U_{e}(x)$ and the free-stream Reynolds number denoted by $R$ are solved to obtain streamwise velocity distribution $u$ and its second derivative $u^{\prime \prime}$. Note that the amplification rate term $\left(-\alpha_{i}\right)$, represents damping if it is negative, a neutrally stable condition of it is zero, and amplification when it is positive.

At a point where the instabilities begin say at $x=x_{1}$, shown in figure 4.1 by point 1 , where the amplification rate term transits from negative to positive, the eigenvalues $\alpha_{r}$ and $\omega$ are computed provided that the local velocity distribution, its second derivative and the local Reynolds number are known. The dimensional frequency computed from equation 4.10 is kept constant along line 1 which is determined by this frequency itself.


Figure 4.1 Schematic of transition calculation using $\mathrm{e}^{\mathrm{n}}$ method

At the next location say $x_{2}$, two separate calculations are performed for the boundary parameters which are newly computed. In one set of calculations the new eigenvalues are computed on the neutral curve as shown by point 2 in the figure using the same procedure explained for the prior station, so that a new dimensional frequency is obtained which defines line 2 . In the second set of calculations, point 1a, the dimensionless frequency $\omega$ together with $L$ and $U_{\infty}$ being known, is determined from the dimensional frequency of line 1 . With $\omega$ and the local Reynolds number of point 2 known, $\alpha$ is obtained using the eigenvalue procedure.

The above explained procedure for point 1 a is then repeated for points 2 b and 1 b respectively and a dimensional frequency is computed for line 3 . For example, at point 1 b , the values $\alpha_{r}$ and $\alpha_{i}$ are computed with the known dimensional frequency of line 1 and the specified local Reynolds number at point 3. At point 2b, they are computed with the known dimensional frequency of line 2 and specified Reynolds number of point 3 .

This procedure is repeated for several number of lines (i.e. frequencies) specified by the user and the variation of the integrated amplification rate defined by $n$ is computed along each line with $x_{0}$ corresponding to the first value of $x$ on the neutral stability curve.


Figure 4.2 Change in integrated amplification factor with respect to distance for Blasius Flow

This procedure leads to curves of constant frequency, with each curve defining the amplification rate as a function of the Reynolds number as shown in figure 4.2 and their envelope corresponds to maximum amplification factors from which transition is computed with a value of $n$ which is commonly assumed between 8 and 9 . Note that in the above figure, the $y$-axis corresponds to amplification factor, where the $x$-axis corresponds to the Reynolds number. As the figure shows Blasius flow, the Reynolds number is defined as square root of local Reynolds number based on the stream-wise distance due to the definition of Blasius length scale.

### 4.2 Numerical Scheme

Since the solution of the Orr-Sommerfeld equation is required in order to predict transition, one has to apply a numerical scheme. The problem that requires attention is
that, since both the equation itself and all the boundary conditions are homogeneous; one faces the problem of obtaining trivial solutions. It is only possible to end up with a nontrivial solution, for some combinations of the Reynolds number, circular frequency and the wave-number. Hence the problem becomes an eigenvalue problem in which values of $R, \alpha$ and $\omega$ are the eigenvalues and the corresponding amplitude functions are eigenfunctions. Note that subscript $l$ in $\alpha$, and bar on top of $\omega$ are dropped for the sake of simplicity. Summarizing, there exists non-trivial solutions of the famous OrrSommerfeld equation. In order to find such a solution, one has to define a relation such as that of equation 4.12
$F(\alpha, \omega, R)=0$

In this study, a finite difference method is employed which is again based on Keller's box scheme. Numerical formulation can be found below recalling again that subscript $l$ in $\alpha$, and bar on top of $\omega$ are dropped for the sake of simplicity.
$\varepsilon_{1}=\alpha^{2}$
$\varepsilon_{2}^{2}=\varepsilon_{1}^{2}+i R(\alpha u-\omega)$
$\varepsilon_{3}=i R \alpha u^{\prime \prime}$

With the introduction of $\varepsilon_{3}$ the Orr-Sommerfeld equation can be written in a compact form as below:
$\phi^{i v}-\varepsilon_{1}^{2} \phi^{\prime \prime}-\varepsilon_{2}^{2}\left(\phi^{\prime \prime}-\varepsilon_{1}^{2} \phi\right)+\varepsilon_{3} \phi=0$

In order to write the above equation as an equivalent system of equations of the first order, new parameters are defined. It is important to distinguish the newly introduced $f$ from the prior one used for the dimensionless stream-function.
$\phi^{\prime}=f$
$f^{\prime}=s+\varepsilon_{1}^{2} \phi$
$s^{\prime}=g$

Hence the above equation 4.14 and its boundary conditions reduce to:
$g^{\prime}=\varepsilon_{2}^{2} s-\varepsilon_{3} \phi$
$y=0 ;$

$$
\begin{equation*}
\{\phi=0, f=0 \tag{4.17a}
\end{equation*}
$$

$y=\frac{\delta}{L} ; \quad\left\{\begin{array}{l}s+\left(\varepsilon_{1}+\varepsilon_{2}\right) f+\varepsilon_{1}\left(\varepsilon_{1}+\varepsilon_{2}\right) \phi=0 \\ g+\varepsilon_{2} s=0\end{array}\right.$

In order to approximate the quantities $(f, s, g, \phi)$, a mesh grid same as that of the boundary layer solution is applied in the $y$-direction, where the edge of the boundary layer is represented by $y_{J}$. As explained in the sections above, equation 4.15 is approximated using a finite-difference method to yield:
$\phi_{j}-\phi_{\phi_{j-1}}-\left(c_{3}\right)_{j}\left(f_{j}+f_{j-1}\right)=\left(r_{1}\right)_{j}=0$
$-f_{j}+f_{j-1}+\left(c_{3}\right)_{j}\left(s_{j}+s_{j-1}\right)+\left(c_{1}\right)_{j}\left(\phi_{j}+\phi_{j-1}\right)=\left(r_{3}\right)_{j-1}=0$
$s_{j}-S_{j-1}-\left(c_{3}\right)_{j}\left(g_{j}+g_{j-1}\right)=\left(r_{2}\right)_{j}=0$
$-g_{j}+g_{j-1}+\left(c_{4}\right)_{j}\left(s_{j}+s_{j-1}\right)+\left(c_{2}\right)_{j}\left(\phi_{j}+\phi_{j-1}\right)=\left(r_{4}\right)_{j-1}=0$

With
$\left(c_{1}\right)_{j}=\varepsilon_{1}^{2}\left(c_{3}\right)_{j}$,
$\left(c_{2}\right)_{j}=-\left(\varepsilon_{3}\right)_{j-1 / 2}\left(c_{3}\right)_{j}$
$\left(c_{3}\right)_{j}=\frac{h_{j-1}}{2}=\frac{y_{j}-y_{j-1}}{2}$,
$\left(c_{4}\right)_{j}=-\left(\varepsilon_{2}^{2}\right)_{j-1 / 2}\left(c_{3}\right)_{j}$

It is important to emphasize that, equations 4.18 are written in such a sequence that the solution matrix $[\underset{\sim}{A}]$ is non-singular, which means it has a non-zero determinant. In addition, note that equations 4.18 are written for all values of $j$ as: $j=1,2, \cdots, J-1$ where to values at $j=0$ and $j=J$ are obtained from the boundary conditions which are given below.
$\phi_{0}=\left(r_{1}\right)_{0}=0, \quad f_{0}=\left(r_{2}\right)_{0}=0$
$f_{J}+\left(\tilde{c}_{3}\right)_{J} s_{J}+\left(\tilde{c}_{1}\right)_{J} \phi_{J}=\left(r_{3}\right)_{J}=0$
$g_{J}+\left(\tilde{c}_{4}\right)_{J} s_{J}=\left(r_{4}\right)_{J}$

Where
$\left(\tilde{c}_{1}\right)_{J}=\varepsilon_{1}$,
$\left(\tilde{c}_{3}\right)_{J}=\frac{1}{\varepsilon_{1}+\varepsilon_{2}}$,
$\left(\tilde{c}_{4}\right)_{J}=\varepsilon_{2}$

Extensive care must be given for the solutions of these equations, since the OrrSommerfeld equation itself and its boundary conditions are homogeneous, therefore trivial solutions such as $f_{j}=s_{j}=\phi_{j}=g_{j}=0$ are valid for all $j$ for all values of $\alpha, \omega$ and $R$. In order to obtain a non-trivial solution, the eigenvalues and eigenfunctions have to be calculated using an iteration procedure to find the values of parameters $\alpha, \omega$ and $R$ to ensure that such a solution exists. Also one has to supply an external nonhomogeneous boundary condition to be satisfied. A clever idea offered by Cebeci et al. is to change the boundary condition $\phi^{\prime}(0)=0\left(\equiv f_{0}=0\right)$ with $\phi^{\prime \prime}(0)=1\left(\equiv s_{0}=1\right)$, so that the difference equations may have a non-trivial solution since $\phi^{\prime \prime}(0) \neq 0$. If one determines the values of parameters using this suggestion, the original boundary condition is satisfied and a non-trivial solution is obtained. This is achieved by an iterative scheme, again based on Newton's method as described in detail in sub-section 3.2.2 using equation 4.18 and wall boundary conditions given below:
$\phi_{0}=\left(r_{1}\right)_{0}=0, \quad s_{0}=\left(r_{2}\right)_{0}=1$

Also the edge boundary conditions given in equation 4.20 are written in matrix form as in equation 3.38.
$[\underset{\sim}{A}] \cdot\{\underset{\sim}{\underset{\sim}{\underset{\sim}{~}}\}}\}=\{\underset{\sim}{\underset{\sim}{r}}\}$

However, in this case the vectors $\{\delta\}$ and $\{r\}$ are different and given in equation 4.23.

$$
\delta_{j}=\left\{\begin{array}{l}
\phi_{j}  \tag{4.23a,b}\\
s_{j} \\
f_{j} \\
g_{j}
\end{array}\right\} \quad r_{j}=\left\{\begin{array}{l}
\left(r_{1}\right)_{j} \\
\left(r_{2}\right)_{j} \\
\left(r_{3}\right)_{j} \\
\left(r_{4}\right)_{j}
\end{array}\right\}
$$

The sub-matrices $A_{j}, B_{j}$ and $C_{j}$, which are coefficients of matrix $[\underset{\sim}{A}]$ (previously $3 \times 3$ but now $4 \times 4$ matrices) are given below in equation 4.24.
$A_{0}=\left[\begin{array}{cccc}1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ \left(c_{1}\right)_{1} & \left(c_{3}\right)_{1} & 1 & 0 \\ \left(c_{2}\right)_{1} & \left(c_{4}\right)_{1} & 0 & 1\end{array}\right]$
$A_{j}=\left[\begin{array}{cccc}1 & 0 & -\left(c_{3}\right)_{j} & 0 \\ 0 & 1 & 0 & \left(c_{3}\right)_{j} \\ \left(c_{1}\right)_{j+1} & \left(c_{3}\right)_{j+1} & 1 & 0 \\ \left(c_{2}\right)_{j+1} & \left(c_{4}\right)_{j+1} & 0 & 1\end{array}\right] \quad 1 \leq j \leq J-1$
$A_{J}=\left[\begin{array}{cccc}1 & 0 & -\left(c_{3}\right)_{J} & 0 \\ 0 & 1 & 0 & -\left(c_{3}\right)_{J} \\ \left(\tilde{c}_{1}\right) & \left(\tilde{c}_{3}\right) & 1 & 0 \\ 0 & \left(\tilde{c}_{4}\right) & 0 & 1\end{array}\right]$

$$
\begin{array}{ll}
B_{j}=\left[\begin{array}{cccc}
-1 & 0 & -\left(c_{3}\right)_{j} & 0 \\
0 & -1 & 0 & -\left(c_{3}\right)_{j} \\
0 & 0 & 0 & 0 \\
0 & 0 & 0 & 0
\end{array}\right] & 1 \leq j \leq J \\
C_{J}=\left[\begin{array}{cccc}
0 & 0 & 0 & 0 \\
0 & 0 & 0 & 0 \\
\left(c_{1}\right)_{j+1} & \left(c_{3}\right)_{j+1} & -1 & 0 \\
\left(c_{3}\right)_{j+1} & \left(c_{4}\right)_{j+1} & 0 & -1
\end{array}\right] & 0 \leq j \leq J-1 \tag{4.24e}
\end{array}
$$

In order to solve equation 3.38, block algorithm is utilized as explained in appendix A. As the solution is also dependent on $\alpha, \omega$ and $R$ as $\underset{\sim}{\delta}=\underset{\sim}{\delta}(\alpha, \omega, R)$ with $\alpha$ complex and $\omega$ real in case of spatial amplification theory implies that the solution of equation 3.38 depends upon 4 scalars. With any two of these fixed, the solution procedure can be carried out to determine the other two in order to satisfy the unmodified boundary conditions.

### 4.3 Calculation of the Neutral Stability Curve and the Dimensional Frequencies

The calculated eigenvalues of the Orr-Sommerfeld equation define whether the disturbances are damped, or amplified as explained in section 4.1. For two dimensional flows, locus defined by $\alpha_{i}=0$ is named the neutral stability curve which separates the amplified region from damped region. If imaginary part of the wave-number is plotted against Reynolds number, the point on the curve with the smallest Reynolds number is of particular interest, since all frequencies prior to this point are stable, and this value is named as the critical Reynolds number.

As described in the above section 4.2, the dimensional frequencies are required in order to use the $e^{n}$ method, and it is found from the solution of the eigenvalue problem in which the solution parameters are the real part of the wave-number and the dimensionless frequency where the Reynolds number and the imaginary part of the wave-number is specified on the neutral curve. Note that this method yields no valid solution if the given Reynolds number is less than the critical Reynolds number. Since in the neutral stability
curve, the complex part of the wave-number is nil, and furthermore the dimensionless frequency is real because of the use of spatial amplification theory, the below given equation 4.25 represents two equations with two unknowns as described at the end of section 4.1, and can be solved using Newton's method in two variables.
$f_{0}(\alpha, \omega, R)=0$

In order to obtain values for the $(k+1)^{\text {th }}$ iteration, the values obtained in the $k^{\text {th }}$ iteration are used as shown below.
$\alpha_{r}^{k+1}=\alpha_{r}^{k}+\delta \alpha_{r}^{k}$
$\omega_{r}^{k+1}=\omega_{r}^{k}+\delta \omega_{r}^{k}$

Expanding equation 4.25 at the $\mathrm{k}^{\text {th }}$ iterate around $\alpha_{r}^{k}$ and $\omega_{r}^{k}$ by Taylor series and neglecting the non-linear terms, one may obtain the following linear system of equations:
$f_{r}^{k}+\left(\frac{\partial f_{r}}{\partial \alpha_{r}}\right) \delta \alpha_{r}^{k}+\left(\frac{\partial f_{r}}{\partial \omega}\right) \delta \omega^{k}=0$
$f_{i}^{k}+\left(\frac{\partial f}{\partial \alpha_{r}}\right) \delta \alpha_{r}^{k}+\left(\frac{\partial f}{\partial \omega}\right) \delta \omega^{k}=0$

And the solution to the above system of linear equations is given by equation 4.28.
$\delta \alpha_{r}^{k}=\frac{1}{\Delta_{0}}\left[f_{i}^{k}\left(\frac{\partial f_{r}}{\partial \omega}\right)^{k}-f_{r}^{k}\left(\frac{\partial f_{r}}{\partial \omega}\right)^{k}\right]$
$\delta \omega_{r}^{k}=\frac{1}{\Delta_{0}}\left[f_{r}^{k}\left(\frac{\partial f_{i}}{\partial \alpha_{r}}\right)^{k}-f_{i}^{k}\left(\frac{\partial f_{r}}{\partial \alpha_{r}}\right)^{k}\right]$

Where $\Delta_{0}$ is the Jacobian of the linear system of equations of equation 4.27, and it is defined by equation 4.28 c.

$$
\begin{equation*}
\Delta_{0}=\left(\frac{\partial f_{r}}{\partial \alpha_{r}}\right)^{k}\left(\frac{\partial f_{i}}{\partial \omega}\right)^{k}-\left(\frac{\partial f_{i}}{\partial \alpha_{r}}\right)^{k}\left(\frac{\partial f_{r}}{\partial \omega}\right)^{k} \tag{4.28c}
\end{equation*}
$$

Equation 3.38 is differentiated with respect to $\alpha_{r}$ and $\omega$ in order to evaluate the derivatives noticed in equation 4.28 c .
$[\underset{\sim}{A}] \cdot\{\underset{\sim}{\delta}\}=\{\underset{\sim}{r}\}$

Noting that the vector $\underset{\sim}{r}$ is independent of our variables, one obtains the following equations named as the variational equations of equation 3.38 with respect to variables $\alpha_{r}$ and $\omega$.
$\underset{\sim}{A}\left(\frac{\partial \underset{\sim}{\delta}}{\partial \alpha_{r}}\right)^{k}=-\left(\frac{\partial \underset{\sim}{A}}{\partial \alpha_{r}}\right)^{k}{\underset{\sim}{\delta}}^{k}$
$\underset{\sim}{A}\left(\frac{\partial \dot{\delta}}{\partial \omega}\right)^{k}=-\left(\frac{\partial A}{\partial \omega}\right)^{k}{\underset{\sim}{\delta}}^{k}$

For the former one, vectors at the RHS are found form equations 4.18, 4.20 and 4.22. Also $\left(r_{1}\right)_{j}=\left(r_{2}\right)_{j}=0$ for $0 \leq j \leq J$ where the other terms of the vector $\underset{\sim}{r}$ are given by equation 4.30 are valid for $1 \leq j \leq J$.
$\left(r_{3}\right)_{j-1}=-2\left(\frac{\partial c_{1}}{\partial \alpha_{r}}\right)_{j} \phi_{j-1 / 2}$
$\left(r_{4}\right)_{j-1}=2\left(\frac{\partial c_{4}}{\partial \alpha_{r}}\right)_{j} s_{j-1 / 2}+2\left(\frac{\partial c_{2}}{\partial \alpha_{r}}\right)_{j} \phi_{j-1 / 2}$
$\left(r_{3}\right)_{J}=-\left(\frac{\partial \tilde{c}_{3}}{\partial \alpha_{r}}\right)_{J} s_{J}-\left(\frac{\partial \tilde{c}_{1}}{\partial \alpha_{r}}\right)_{J} \phi_{J}$
$\left(r_{4}\right)_{J}=-\left(\frac{\partial \tilde{c}_{4}}{\partial \alpha_{r}}\right)_{J} s_{J}$

For the latter one, that is equation 4.28b, again $\left(r_{1}\right)_{j}=\left(r_{2}\right)_{j}=0$ for $0 \leq j \leq J$ where the other terms of the vector $\underset{\sim}{r}$ are given by equation 4.31 valid for $1 \leq j \leq J$.
$\left(r_{3}\right)_{j-1}=0$
$\left(r_{4}\right)_{j-1}=-2\left(\frac{\partial c_{4}}{\partial \omega}\right)_{j} s_{j-1 / 2}$
$\left(r_{3}\right)_{J}=\left(\frac{\partial \tilde{c}_{3}}{\partial \omega}\right)_{J} s_{J}$
$\left(r_{4}\right)_{J}=-\left(\frac{\partial \tilde{c}_{4}}{\partial \omega}\right)_{J} S_{J}$

Using assumed values of $\alpha_{r}$ and $\omega$, at a fixed $R$, if equation 4.25 is satisfied, and also to a good approximation if $f_{0}=0$, one may ensure that assumed values are good enough so that no further computation of $\alpha_{r}$ and $\omega$ is required. But if $f_{0} \neq 0$, one has to compute new values of $\alpha_{r}$ and $\omega$ using Newton's method. After solving equation 4.29, one may obtain the derivatives of $f$ with respect to $\alpha_{r}$ and $\omega$. Using this new information, equation 4.28 may be solved. The iteration process is repeated until the values of parameters in equation 4.28 are less than a specified value to check for convergence.

After a valid solution for equation 4.25 is obtained, one seeks for the values for the derivatives of $\alpha_{r}$ and $\omega$ with respect to Reynolds number at a specific location $R=R_{0}$. One gets the following equation after taking the total derivative of equation 4.25.
$\left(\frac{\partial f_{r}}{\partial \alpha_{r}}\right)_{0}\left(\frac{\partial \alpha_{r}}{\partial R}\right)_{0}+\left(\frac{\partial f_{r}}{\partial \omega}\right)_{0}\left(\frac{\partial \omega}{\partial R}\right)_{0}=-\left(\frac{\partial f_{r}}{\partial R}\right)_{0}$

$$
\begin{equation*}
\left(\frac{\partial f_{i}}{\partial \alpha_{r}}\right)_{0}\left(\frac{\partial \alpha_{r}}{\partial R}\right)_{0}+\left(\frac{\partial f_{i}}{\partial \omega}\right)_{0}\left(\frac{\partial \omega}{\partial R}\right)_{0}=-\left(\frac{\partial f_{i}}{\partial R}\right)_{0} \tag{4.32b}
\end{equation*}
$$

In addition,

$$
\begin{align*}
& \left(\frac{\partial \alpha_{r}}{\partial R}\right)_{0}=\frac{1}{\Delta_{0}}\left[\left(\frac{\partial f_{i}}{\partial R}\right)_{0}\left(\frac{\partial f_{r}}{\partial \omega}\right)_{0}-\left(\frac{\partial f_{r}}{\partial R}\right)_{0}\left(\frac{\partial f_{i}}{\partial \omega}\right)_{0}\right]  \tag{4.33a}\\
& \left(\frac{\partial \omega}{\partial R}\right)_{0}=\frac{1}{\Delta_{0}}\left[\left(\frac{\partial f_{r}}{\partial R}\right)_{0}\left(\frac{\partial f_{i}}{\partial \alpha_{r}}\right)_{0}-\left(\frac{\partial f_{i}}{\partial R}\right)_{0}\left(\frac{\partial f_{r}}{\partial \alpha_{r}}\right)_{0}\right] \tag{4.33b}
\end{align*}
$$

Where the term $\Delta_{0}$ is given in equation 4.33c.

$$
\begin{equation*}
\Delta_{0}=\left(\frac{\partial f_{r}}{\partial \alpha_{r}}\right)_{0}\left(\frac{\partial f_{i}}{\partial \omega}\right)_{0}-\left(\frac{\partial f_{r}}{\partial \omega}\right)_{0}\left(\frac{\partial f_{i}}{\partial \alpha_{r}}\right)_{0} \tag{4.33c}
\end{equation*}
$$

In order to evaluate the derivatives with respect to Reynolds number in equation 4.33, variational equations with respect to Reynolds number are written in equation 4.34.

$$
\begin{equation*}
A\left(\frac{\partial{\underset{\sim}{\delta}}^{k}}{\partial R}\right)=-\left(\frac{\partial \underset{\sim}{A}}{\partial R}\right)^{k} \underset{\sim}{\delta} \tag{4.34}
\end{equation*}
$$

Vectors at the RHS are $\left(r_{1}\right)_{j}=\left(r_{2}\right)_{j}=0$ for $0 \leq j \leq J$ where the other terms of the vector $\underset{\sim}{r}$ are given by equation 4.35 valid for $1 \leq j \leq J$.
$\left(r_{3}\right)_{j-1}=0$
$\left(r_{4}\right)_{j-1}=2\left(\frac{\partial c_{4}}{\partial R}\right) s_{j-1 / 2}+2\left(\frac{\partial c_{2}}{\partial R}\right) \phi_{j-1 / 2}$
$\left(r_{3}\right)_{J}=-\left(\frac{\partial \tilde{c}_{3}}{\partial R}\right)_{J} s_{J}$

$$
\begin{equation*}
\left(r_{4}\right)_{J}=-\left(\frac{\partial \tilde{c}_{4}}{\partial R}\right)_{J} s_{J} \tag{4.35d}
\end{equation*}
$$

Derivatives of $f$ with respect to $R$ are obtained from the solution of the variational equation with respect to Reynolds number.

### 4.4 Calculating the Onset of Transition

In order to calculate the onset of transition, a slightly different eigenvalue problem has to be imposed. In the case of calculating neutral stability, the methodology used is to set the imaginary part of the wave-number as zero at a fixed position that is a local Reynolds number and solve for the other two unknown eigenvalues. However, this case differs from the previous one at the point where the two eigenvalues are input in order to solve for the other two. In the previous case, the frequencies are determined, and therefore value of $\omega$ is known. The procedure is to check each point until a station where transition is found. Therefore, two unknown eigenvalues are needed to be found for fixed $\omega$ and $R$.

Expanding equation 4.25 at the $\mathrm{k}^{\text {th }}$ iterate again using Taylor series, but this time with other variables, as our problem is different, and furthermore neglecting the non-linear terms, one obtains:

$$
\begin{align*}
& f_{r}^{k}+\left(\frac{\partial f_{r}}{\partial \alpha_{r}}\right) \delta \alpha_{r}^{k}+\left(\frac{\partial f_{r}}{\partial \alpha_{i}}\right) \delta \alpha_{i}^{k}=0  \tag{4.36a}\\
& f_{i}^{k}+\left(\frac{\partial f_{i}}{\partial \alpha_{r}}\right) \delta \alpha_{r}^{k}+\left(\frac{\partial f_{i}}{\partial \alpha_{i}}\right) \delta \alpha_{i}^{k}=0 \tag{4.36b}
\end{align*}
$$

Also the solution to the above set of linear equations is very similar to those of equation 4.27.

$$
\begin{equation*}
\delta \alpha_{r}^{k}=\frac{1}{\Delta_{0}}\left[f_{i}^{k}\left(\frac{\partial f_{r}}{\partial \alpha_{i}}\right)^{k}-f_{r}^{k}\left(\frac{\partial f_{r}}{\partial \alpha_{i}}\right)^{k}\right] \tag{4.37a}
\end{equation*}
$$

$\delta \alpha_{i}^{k}=\frac{1}{\Delta_{0}}\left[f_{r}^{k}\left(\frac{\partial f_{i}}{\partial \alpha_{r}}\right)^{k}-f_{i}^{k}\left(\frac{\partial f_{r}}{\partial \alpha_{r}}\right)^{k}\right]$

Where $\Delta_{0}$ is given in equation 4.37 c
$\Delta_{0}=\left(\frac{\partial f_{r}}{\partial \alpha_{r}}\right)^{k}\left(\frac{\partial f_{i}}{\partial \alpha_{i}}\right)^{k}-\left(\frac{\partial f_{i}}{\partial \alpha_{r}}\right)^{k}\left(\frac{\partial f_{r}}{\partial \alpha_{i}}\right)^{k}$

This time, equation 3.38 is differentiated with respect to $\alpha$ in order to evaluate the derivatives noticed in equation 4.37 c .
$[\underset{\sim}{A}] \cdot\{\underset{\sim}{\delta}\}=\{\underset{\sim}{r}\}$

Again the vectors at the RHS are found form equations 4.18, 4.20 and 4.22. $\left(r_{1}\right)_{j}=\left(r_{2}\right)_{j}=0$ for $0 \leq j \leq J$ where the other terms of the vector $\underset{\sim}{r}$ are given again by equation 4.30 valid for $1 \leq j \leq J$. Note that the $)_{r}$ terms in coefficients must be dropped, since the differentiation is not done with respect to $\alpha_{r}$, but for $\alpha\left(\equiv \alpha_{r}+i \alpha_{i}\right)$ itself.

## CHAPTER 5

## TURBULENCE MODEL

### 5.1 Turbulence Model Used for the Boundary Layer

Since the viscosity term $v$ is replaced by a $b$ as defined in equation 2.18 , which includes the eddy viscosity term, the equations written are valid for both laminar and turbulent flows. However, this brings the requirement to introduce an appropriate definition of the eddy viscosity term, $\varepsilon_{m}$. This is achieved by introduction of a turbulence model, which in this study is selected as the Cebeci and Smith algebraic turbulence model. The model used was first introduced by Cebeci and Smith [21]. This method is feasible and it is shown that for external flows, use of higher order models does not bring much more accuracy [22, 23].

First person to assume that turbulent stresses act like laminar stresses was Boussineque. This implies the fact that turbulent stresses also can be explained with respect to the velocity gradient. In this case, the proportionality constant is named as the eddy viscosity and is defined below in equation 2.17.
$-\overline{\hat{u} \hat{\nu}}=\varepsilon_{m} \frac{\partial u}{\partial y}$

In the above equation, eddy viscosity is assumed to be a product of a length and a velocity. The trick is to select the proper length and velocity scales for turbulent flow. The mixing length concept follows this fact. In mixing length theory, motions of fluid eddies are considered to be analogous to motions of the free gas molecules. It is assumed that, after the fluid eddies travel for some distance they breakdown and new eddies are formed from those which have broken down. The distance traveled in transverse direction is named the mixing length denoted by $l$ and the velocity at which they move is proportional to the fluctuation velocity. Therefore the eddy viscosity is defined as shown in equation 5.1.

$$
\begin{equation*}
\varepsilon_{m}=\hat{v} l \tag{5.1}
\end{equation*}
$$

Generalizing this idea, Prandtl has introduced a mixing length theory with the following formulation:

$$
\begin{equation*}
-\overline{\hat{u} \hat{v}}=l^{2}\left|\frac{\partial u}{\partial y}\right| \frac{\partial u}{\partial y} \tag{5.2}
\end{equation*}
$$

As described in [21], the fluctuation velocity $v^{\prime}$ is approximated by the following relation:

$$
\begin{equation*}
v^{\prime} \sim l\left(\frac{\partial u}{\partial y}\right) \tag{5.3}
\end{equation*}
$$

So that equation 5.1 can be written as:

$$
\begin{equation*}
\varepsilon_{m}=l^{2}\left|\frac{\partial u}{\partial y}\right| \tag{5.4}
\end{equation*}
$$

According to van Driest, the mixing length is:

$$
\begin{equation*}
l=0.4 y\left[1-\exp \left(-\frac{y}{A}\right)\right] \tag{5.5}
\end{equation*}
$$

Where $A$ is the damping factor defined in equation 5.6.

$$
\begin{equation*}
A=26 v u_{\tau}^{-1} \tag{5.6}
\end{equation*}
$$

With newly introduced terms defined in equation 5.7.

$$
\begin{equation*}
u_{\tau}=\left(\frac{\tau_{l a m}}{\rho}\right)_{\max }^{1 / 2} \quad \tau_{l a m}=\mu \frac{\partial u}{\partial y} \tag{5.7a,b}
\end{equation*}
$$

It is of particular importance that in a turbulent boundary layer as $y \rightarrow 0$, that is as the laminar sub-layer is approached, the mixing length approaches zero and hence the eddy viscosity vanishes. Therefore, the expression of eddy viscosity given above is only
acceptable at the inner layer, wherein according to Klebanoff, both mixing length and the eddy viscosity are linear functions of the distance from the wall, which is $y$.

As departed from the wall that is in the outer layer, non-linear effects gain higher importance for the eddy viscosity, and the eddy viscosity decreases slowly according to Klebanoff. On the other hand, the mixing length remains the same, and according to [21], the eddy viscosity in the outer layer may be given as:
$\varepsilon_{m}=\alpha U_{e} \delta^{*}$

In the above formulation, the factor $\alpha$ is a coefficient defined as:

$$
\begin{array}{ll}
\alpha=0.0168 & \text { for } R_{\theta} \geq 5000 \\
\alpha=0.0168 \frac{1.55}{1+\pi} & \text { elsewhere } \tag{5.9b}
\end{array}
$$

Where the term $\pi$ is called the Cole's wake function and given by equation 5.10.
$\pi=0.55\left[1-e^{\left(-0.342 z_{1}^{1 / 2}-0.298 z_{1}\right)}\right]$

Where

$$
\begin{equation*}
z_{1}=\frac{R_{\theta}}{425}-1 \quad \text { for } R_{\theta}>425 \tag{5.11}
\end{equation*}
$$

It is literally impossible to determine whether the edge of the boundary layer is reached or not, since it is hard to distinguish the very outer end of the boundary layer from the freestream. In order to eliminate problems related to this situation, Klebanoff introduced a so called intermittency factor as shown in equation 5.12.
$\gamma=\left[1+5.5\left(\frac{y}{\delta}\right)^{6}\right]^{-1}$

And using the above factor, equation 5.8 is rewritten as equation 5.13.
$\varepsilon_{m}=\alpha U_{e} \delta^{*} \gamma$

Therefore, with $y_{c}$ denoting the point at which inner layer is separated from the outer layer, the eddy viscosity is given by equation 5.14. The definition of $y_{c}$ is that it is the point at which the inner and outer eddy viscosity formulations gives the same result.

$$
\varepsilon_{m}=\left\{\begin{array}{cc}
l^{2}\left|\frac{\partial u}{\partial y}\right| \gamma_{t r} & \left(0 \leq y \leq y_{c}\right)  \tag{5.14}\\
\alpha\left|\int_{0}^{\infty}\left(U_{e}-u\right) d y\right| \gamma_{t r} \gamma & \left(y_{c} \leq y \leq \delta\right)
\end{array}\right\}
$$

Where the term $\gamma_{t r}$ is the factor which accounts for smooth transition from laminar to turbulent flow and it is given by equation 5.15.

$$
\begin{equation*}
\gamma_{t r}=1-\exp \left[-G\left(x-x_{t r}\right) \int_{x_{r r}}^{x} \frac{d x}{U_{e}}\right] \quad \text { with } \quad G=\frac{2}{C^{2}} \frac{U_{e}^{3}}{v^{2}} R_{x_{r r}}^{-1.34} \tag{5.15,16}
\end{equation*}
$$

In the above equation, the term $R_{x_{t, t}}$ is the transition Reynolds number. The parameter $C$ is a coefficient which is given below:

$$
\begin{array}{ll}
C=60 & \text { for } R_{c}>4.5 \times 10^{6} \\
C^{2}=213\left[\log R_{x_{t r}}-4.7323\right] & \text { for } R_{c}<4.5 \times 10^{6}
\end{array}
$$

For the case of flows with strong pressure gradient, the value of coefficient alpha has to be a variable dependent on $R_{\theta}$ even if it is larger than 5000 . In order to take this case into account, Simpson's suggestion as described in [24] is used. For further information refer to reference [25] also. Simpson's suggestion relates to value of $\alpha$ to $F$; which represents the ratio of the turbulent energy multiplied by normal stresses to the turbulent energy multiplied by shear stress, evaluated where the shear stress is maximum.

$$
\begin{align*}
& \alpha=\frac{0.0168}{F^{2.5}}  \tag{5.18}\\
& F=\left\{\frac{\left(\overline{\hat{u}^{2}}-\overline{\hat{v}^{2}}\right) \frac{\partial u}{\partial x}}{-\overline{\hat{u} \hat{v}} \frac{\partial u}{\partial y}}\right\}_{\left((\overline{\bar{u}})_{\max }\right.} \tag{5.19}
\end{align*}
$$

Also defining a parameter $\beta$ as:

$$
\begin{equation*}
\beta=\left\{\frac{\overline{\bar{u}^{2}}-\overline{\hat{v}^{2}}}{-\overline{\hat{u} \hat{v}}}\right\}_{(-\bar{u} \bar{v})_{\max }} \tag{5.20}
\end{equation*}
$$

Which according to Nakayama as referenced in [26] is expressed as a function of $R_{T}$ defined in equation 5.21.

$$
\begin{equation*}
R_{T}=\frac{\tau_{w}}{(-\overline{\hat{u} \hat{v}})_{\max }} \tag{5.21}
\end{equation*}
$$

So that $\beta$ becomes:

$$
\begin{array}{ll}
\beta=\frac{6}{1+2 R_{T}\left(2-R_{T}\right)} & \text { for } R_{T}<1.0  \tag{5.22a}\\
\beta=\frac{2 R_{T}}{1+R_{T}} & \text { for } R_{T} \geq 1.0
\end{array}
$$

Combining all these relations in a single one written for $\alpha$, one gets equation 5.23.
$\alpha=\frac{0.0168}{\left[1-\beta\left(\frac{\partial u}{\partial x}\right)\left(\frac{\partial u}{\partial y}\right)^{-1}\right]^{2.5}}$

### 5.2 Turbulence Model Used for the Wake Region

Like the boundary layer equations for the wake region, the turbulence model for the wake region has to be modified since the wake is not a wall bounded flow. Also analogous to the change of boundary conditions from no-slip conditions to smooth flow conditions, the change on the eddy viscosity must be applied gradually. The eddy viscosity shall be close to that of the wall near the trailing edge, and gradually approach the far-wake value as proceeded in the downstream direction. Chang et al. suggests applying the following relation as referenced in [22].
$\varepsilon_{m}=v_{w}+\left(v_{\text {T.E. }}-v_{w}\right) e^{-B_{1}}$

Where in the above formulation, $v_{\text {T.E. }}$ represents the value of eddy viscosity at the trailing edge. The term $v_{w}$ stands for value of the eddy viscosity at the far wake region, and has it maximum values for the upper and lower parts of the wake defined as shown in equation 5.25. The location $y_{\text {min }}$ corresponds to the point on the velocity profile with minimum velocity value.

$$
\left.\begin{array}{l}
v_{w_{l}}=0.064 \int_{-\infty}^{y_{\text {min }}}\left(U_{e}-u\right) d y  \tag{5.25}\\
v_{w_{u}}=0.064 \int_{y_{\text {min }}}^{\infty}\left(U_{e}-u\right) d y
\end{array}\right\}
$$

The coefficient $B_{1}$ is defined as shown below in equation 5.26.
$B_{1}=\frac{\left(x-x_{T . E .}\right)}{20 \delta_{\text {T.E. }}}$

## CHAPTER 6

## RESULTS AND DISCUSSION

Theory explained in Chapters 3, 4 and 5 are applied to a computer program suite written in Fortran 77 programming language. The IBLWEN Suite is combining interactive boundary layer methods with the Smith-van Ingen Method for prediction of the onset of transition. The stability transition method is first demonstrated on a flat plate with Blasius and Falkner Skan flows. Following, the whole suite is tested on two different airfoils up to stall angles of attack. Among these, the first one is the NACA 0012 Airfoil, and the second one is the NACA 4412 airfoil. Two airfoils have the same thickness ratio of $12 \%$ with the same thickness distribution; and the only difference is that the latter one has a camber ration of 4 percent. It shall also be noted that compressibility effects are neglected, which limits the use of the method prescribed to Mach numbers essentially less than 0.3 , where errors due to compressibility effects are acceptable.

In order to obtain external velocity distribution and the stagnation point for the airfoil supplied, the Hess-Smith panel method is employed. Preceding that, the boundary layer method is employed to solve for the upper and lower surfaces starting from the stagnation point up to the trailing edge.

Location for the transition from laminar to turbulent flow is either input, or calculated using the $\mathrm{e}^{\mathrm{n}}$ method. When the Reynolds number is high, and angles of attack are low to moderate, the onset of transition occurs before the separation point, and the extent of the transition region is confined to a relatively small region. Sometimes, however before transition location can be computed with the procedure explained, laminar separation takes place. In that case the separation location may be assumed to correspond to the onset of transition as stated in reference [5]. Therefore, if no transition is found until a laminar separation occurs, this point is assumed to be the point of laminar to turbulent transition.

In the $e^{n}$ method, laminar velocity profiles are analyzed with the linear stability approach and the amplification rates are computed from the physical frequencies of the instability
waves as described in Chapter 4. When the n-factor, that is the integrated amplification rate, reaches a value between 8 and 10 upon preference of the user, transition is assumed to be found.

After the onset of transition is determined, the value is supplied to the program suite externally and inviscid-viscous calculations are carried on to obtain the flow field around the airfoil. Given an external velocity distribution, each boundary layer solution starting form the stagnation point and ending in the wake region is called a sweep. After a number of sweeps are accomplished, the blowing velocity distribution is found and is used to obtain a new external velocity distribution using the Hess-Smith panel method. This whole procedure including the updating of the external velocity distribution and doing some number of sweeps is called a cycle. One must set the number of cycles to a high enough value to ensure convergence.


Figure 6.1 Convergence history for NACA 0012 Airfoil $\left(R_{c}=6 \times 10^{6}, \alpha=12^{\circ}\right)$

While calculating the potential flow around the airfoil at high angles of attack (i.e. when a separated zone exists near the trailing edge, one has to evaluate the velocities and supply the Kutta condition at the edge of the displacement thickness as stated in references [1] and [3]. On the other hand, even if this is the case, the blowing velocity distribution obtained from the boundary layer solutions are applied on to the airfoil surface as boundary conditions. An over relaxation scheme which is taken from the work done by Özgen is applied in order to accelerate convergence. A sample convergence history plot is given in figure 6.1 . As seen in figure 6.1 , after 20 cycles, one obtains the final converged result.

### 6.1 Falkner Skan Flow

In order to demonstrate the transition prediction procedure, the $e^{n}$ program was first run using Falkner-Skan velocity profiles, for which after transformation of variables, velocity profile is the same for all stations. Calculations are carried out for Falkner-Skan parameter $\beta$ which is defined in equation 6.1, ranging from -0.1988 to 0.5 . Note that the parameter $m$ seen in equation 6.1 is the pressure gradient parameter.
$\beta=\frac{2 m}{m+1}$
$m=\frac{x}{U_{e}} \frac{d U_{e}}{d x}$

Neutral stability calculations agree very well with Özgen and Mack as in figure 6.2 given for Blasius flow, which is a Falkner-Skan flow with no pressure gradient. Figures from 6.3 to 6.13 are the integrated amplification factor, $n$, plotted against the local Reynolds number. It shall once again be emphasized that the local Reynolds number used is based on the Falkner-Skan length scale, which is the square root of actual (i.e. non-transformed) Reynolds number. One may easily notice that the transition location moves downstream as the pressure gradient parameter increases, that is pressure gradient changes from adverse to favorable. Values of the parameter $\beta$ indicate an adverse pressure gradient if it is less than zero, no gradient if it is equal to zero and a favorable pressure gradient if it is greater than zero.


Figure 6.2 Neutral Stability Curve for Blasius Flow


Figure 6.3 Change in integrated amplification factor with respect to distance for Falkner Skan Flow with $\beta=-0.1988$


Figure 6.4 Change in integrated amplification factor with respect to distance for Falkner Skan Flow with $\beta=-0.15$


Figure 6.5 Change in integrated amplification factor with respect to distance for Falkner
Skan Flow with $\beta=-0.1$


Figure 6.6 Change in integrated amplification factor with respect to distance for Falkner Skan Flow with $\beta=-0.05$


Figure 6.7 Change in integrated amplification factor with respect to distance for Blasius Flow


Figure 6.8 Change in integrated amplification factor with respect to distance for Falkner Skan Flow with $\beta=0.05$


Figure 6.9 Change in integrated amplification factor with respect to distance for Falkner Skan Flow with $\beta=0.1$


Figure 6.10 Change in integrated amplification factor with respect to distance for Falkner Skan Flow with $\beta=0.2$


Figure 6.11 Change in integrated amplification factor with respect to distance for Falkner Skan Flow with $\beta=0.3$


Figure 6.12 Change in integrated amplification factor with respect to distance for Falkner Skan Flow with $\beta=0.4$


Figure 6.13 Change in integrated amplification factor with respect to distance for Falkner Skan Flow with $\beta=0.5$

### 6.2 Airfoil Flow

The program suite developed is tested on two airfoils as mentioned above. The first one tested is the NACA 0012 airfoil, which is $12 \%$ thick, with maximum thickness located at $30 \%$, and having no camber. The second airfoil is the NACA 4412 airfoil again $12 \%$ thick with maximum thickness located at $40 \%$, plus having $4 \%$ camber ratio located at $30 \%$ chord wise distance.

While performing calculations around an airfoil, provided that there is no or very little separated zone on the airfoil, wake calculations can be neglected on choice. Since no harm to overall result is made, wake region is included in all calculations done. As emphasized in reference [28], since the viscous effects are introduced into the panel method through the blowing velocity distribution and off-body Kutta condition, both of which involve the displacement thickness, the accuracy of the inviscid flow depends on the accuracy of determining the displacement thickness distribution everywhere in the flow field, especially at the airfoil trailing edge and the wake.


Figure 6.14 Variation of lift coefficient with respect to angle of attack for NACA 0012 airfoil for $R_{c}=2 \times 10^{5}$ and $R_{c}=6 \times 10^{6}$

Figure 6.14 shows coefficient of lift against angle of attack for NACA 0012 airfoil. Results obtained at $R_{c}=6 \times 10^{6}$ for coefficient of lift is in perfect accord with measurements, although post stall solutions can not be obtained due to growth of the separated region. Even there is no experimental data found for $R_{c}=2 \times 10^{5}$, it can be told that the code developed accounts for the Reynolds number effects acceptably. As seen in figure 6.14, Reynolds number affects both the maximum coefficient of lift and the stall attack significantly.

Prediction of transition location is important in calculation of the flow field and therefore performance characteristics of an airfoil. The $e^{n}$ method gives very good results, but may only be used from low to moderate angles of attack. At high angles of attack, the transition location moves upstream and it is not possible to find transition in the attached laminar zone. Therefore, the laminar separation point is assumed to be the transition location. In table 6.1, transition locations found on the upper surface of NACA 0012 airfoil at chord Reynolds numbers and three angles-of-attack are compared to those calculated by Cebeci et al from reference [28]. Figure 6.15 shows the amplification rates as a function of chord-wise distance from the leading edge for three physical frequencies for the lower surface of NACA 0012 airfoil at $\alpha=4^{\circ}$ for a chord wise Reynolds number of $6 \times 10^{6}$. For this case, the transition is found using the $e^{n}$ method at $53.28 \%$ of the chord.

Table 6.1 Comparison of transition locations on the upper surface of a NACA 0012 Airfoil at two chord Reynolds numbers and three angles-of-attack.

|  | $\alpha=0^{\circ}$ |  | $\alpha=2^{\circ}$ |  | $\alpha=4^{\circ}$ |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Calculated | Cebecietal | Calculated | Cebeciet al | Calculated | Cebeciet al |
| $R_{c}$ | $(s /)_{1 r}-(x /)^{\prime \prime}$ | $(s /)_{r r}-(x /)_{)_{r r}}$ | $(s /)_{w r}-(x /)_{r r}$ | $(s /)_{r r}-(x /)_{)_{w}}$ | $(s /)_{)_{r}}-(x /)_{w_{r}}$ | $\left(s / c_{t r}-(x /)_{)_{r r}}\right.$ |
| $1 \times 10^{6}$ | 0.502-0.486 | 0.505-0.49 | 0.329-0.307 | 0.33-0.31 | 0.168-0.141 | 0.16-0.13 |
| $3 \times 10^{6}$ | 0.353-0.336 | 0.355-0.34 | 0.206-0.184 | 0.21-0.19 | 0.107-0.08 | 0.10-0.075 |



Figure 6.15 Amplification rates calculated using the $\mathrm{e}^{\mathrm{n}}$ method for lower surface of NACA 0012 Airfoil at $R_{c}=6 \times 10^{6}$ and $\alpha=4^{\circ}$.

Figure 6.16 shows the variation of drag coefficient with respect to lift coefficient. Drag is estimated using the formulation by Squire and Young [24] which is given below in equation 6.3. In equation 6.5 the same formula is given in another form.

$$
\begin{align*}
& C_{d}=\frac{2 \theta_{T . E .}}{c}\left(\frac{u_{T . E .}}{U_{\infty}}\right)^{\frac{H_{T . E .}+5}{2}} \\
& H_{T . E .}=\frac{\delta_{T . E .}^{*}}{\theta_{\text {T.E. }}}  \tag{6.4}\\
& C_{d}=\frac{2 \theta_{\infty}}{c} \tag{6.4}
\end{align*}
$$

In equation the terms $\theta_{T . E}$ and $u_{\text {T.E. }}$ are the momentum thickness and the edge velocity at the trailing edge respectively. Free stream velocity is $U_{\infty}$, while $H_{\text {T.E }}$ is the shape factor at the trailing edge and is defined in equation 6.4. In equation 6.5 , the term $\theta_{\infty}$ denotes the momentum thickness at the far wake.


Figure 6.16 Drag polar for NACA 0012 airfoil for $R_{c}=2 \times 10^{5}$ and $R_{c}=6 \times 10^{6}$
As seen in figure 6.16, the estimated drag value agrees well with the measurements except for slight underestimation at moderate to high lift values. The maximum deviation is observed at a lift coefficient value of 1.25 as $12.5 \%$.

Given in figure 6.17 is the variation of displacement thickness at the trailing edge with respect to the angle of attack. Notice that slope of the curve increases with increasing angle of attack. At 17 degrees, the displacement thickness value reaches approximately about $6 \%$ of the chord.

Figure 6.18 shows the viscous and inviscid pressure distribution. Since the area enclosed by inviscid pressure distribution is slightly higher than the area enclosed by the viscous one, the inviscid method overestimates the lift coefficient, as expected.

In figure 6.19, the displacement thickness distribution on the upper surface of the NACA 0012 airfoil at $\alpha=10^{\circ}$ is shown. The displacement thickness first rises in the stream wise direction until trailing edge and then falls as going towards the far wake.


Figure 6.17 Variation of the displacement thickness with respect to angle of attack at the trailing edge for NACA 0012 airfoil for $R_{c}=6 \times 10^{6}$


Figure 6.18 Inviscid and viscous pressure distribution around NACA 0012 airfoil for

$$
R_{c}=6 \times 10^{6} \text { at } \alpha=14^{\circ}
$$



Figure 6.19 Displacement thickness distribution at the upper surface of NACA 0012 airfoil and its wake for $R_{c}=6 \times 10^{6}$ and $\alpha=10^{\circ}$

Figure 6.20 shows velocity profiles at some selected stations. As seen in the figure, flow accelerates until approximately $6 \%$ of the chord and the starts to decelerate. As expected, the boundary layer thickness also increases traversing in the downstream direction.


Figure 6.20 Selected velocity profiles at the upper surface of NACA 0012 airfoil for $R_{c}=6 \times 10^{6}$ and $\alpha=15^{\circ}$

In figures 6.21 to 6.23 , velocity profiles in the wake region are shown for NACA 0012 airfoil at $R_{c}=6 \times 10^{6}$ with angles of attack values of $0^{\circ}, 8^{\circ}$ and $15^{\circ}$ respectively.

Velocity profiles for $\alpha=0^{\circ}$ shown in figure 6.21 are perfectly symmetrical as expected. One other important result is that the high velocity gradients no longer exist in the far wake and are replaced by smooth gradients. At 3 chords downstream from the trailing edge, where the calculations end, maximum deviation from the free stream velocity is found to be approximately $5 \%$. In figure 6.22 , which is for $\alpha=8^{\circ}$, it is not possible to mention about symmetry but the far wake velocity profiles are very close to the free stream profile as expected.

Figure 6.23 shows the wake velocity profiles for $\alpha=15^{\circ}$. In this figure, flow at the lower surface is laminar where the upper surface is turbulent. Moreover, the last $10 \%$ of the flow is separated. Therefore the thickness of the boundary layer coming from the upper surface is approximately 20 times larger than that of the lower surface. As desired, the two velocity profiles merge and result in a free stream like velocity profile at the far wake.


Figure 6.21 Velocity profiles in the wake for NACA 0012 airfoil for $R_{c}=6 \times 10^{6}$ and

$$
\alpha=0^{\circ}
$$



Figure 6.22 Velocity profiles in the wake for NACA 0012 airfoil for $R_{c}=6 \times 10^{6}$ and $\alpha=8^{\circ}$


Figure 6.23 Velocity profiles in the wake for NACA 0012 airfoil for $R_{c}=6 \times 10^{6}$ and $\alpha=15^{\circ}$

Given in figure 6.24 is the variation of surface friction coefficient with respect to distance traversed in downstream direction measured from the stagnation point for upper surface of the NACA 0012 airfoil with Reynolds number based on chord of $2 \times 10^{5}$. The jumps in $C_{f}$ values show transition. $C_{f}$ values less than zero show separation. For $\alpha=12^{\circ}$, approximately $26 \%$ of the flow is separated. Also one may notice the laminar separation taking place at $s / c=0.82$ for $\alpha=0^{\circ}$


Figure 6.24 Variation of drag surface friction coefficient against surface distance traversed for NACA 0012 airfoil for $R_{c}=2 \times 10^{5}$

The other airfoil tested is the NACA 4412 airfoil with $4 \%$ of camber. The test were carried out for $R_{c}=6 \times 10^{6}$ and for $R_{c}=2 \times 10^{5}$. Lift coefficient with respect to angle of attack is plotted in figure 6.25. Again, the results obtained are in good accord with measurements. Like in NACA 0012 airfoil, large separated regions on the upper surface at high angles of attack prevent to find solutions for post stall cases.

Figure 6.26 shows variation of drag coefficient with respect to lift coefficient. For moderate values of lift coefficient, a slight overestimation of drag coefficient is observed. On the other hand for low and high values the values are slightly underestimated. Overall, the drag polar is within acceptable values. Maximum deviation between measured calculated values of drag coefficient is found to be $15 \%$.


Figure 6.25 Variation of lift coefficient with respect to angle of attack for NACA 4412 airfoil for $R_{c}=2 \times 10^{5}$ and $R_{c}=6 \times 10^{6}$

Given in figure 6.27 is the variation of surface friction coefficient with respect to distance traversed in downstream direction measured from the stagnation point for upper surface of the NACA 4412 airfoil with Reynolds number based on chord of $2 \times 10^{5}$. The jumps in $C_{f}$ values show transition. $C_{f}$ values less than zero show separation. For $\alpha=12^{\circ}$, approximately $32 \%$ of the flow is separated. Also one may notice the laminar separation taking place at $s / c=0.16$ for $\alpha=12^{\circ}$


Figure 6.26 Drag polar for NACA 4412 airfoil for $R_{c}=2 \times 10^{5}$ and $R_{c}=6 \times 10^{6}$


Figure 6.27 Variation of drag surface friction coefficient against surface distance traversed for NACA 4412 airfoil for $R_{c}=2 \times 10^{5}$

## CHAPTER 7

## CONCLUSION

In this study, Smith-van Ingen transition prediction procedure is implemented into the previous study carried out by Özgen. The resultant program suite is tested for Falkner Skan flows and two airfoils. For airfoil solutions, the results are in good accord with measurements until the stall angle of attack.

At high angles of attack, due the growth of the boundary layer and the displacement thickness, constant pressure assumption across the boundary layer becomes less accurate. Due to this reason and excessive separated regions in the flow, post stall solutions can not be performed.

In this study, Cebeci-Smith algebraic turbulence model is used. Any empirical model which formulated the turbulent viscosity may easily be implemented to the computer program. On the other hand, in order to solve for largely separated flows, employment of a higher order turbulence model may be more suitable.

As stated in reference [1], determination of transition plays a key role in obtaining correct solutions. Implementing a more concrete physical means to calculate transition was aimed in this study. For moderate Reynolds numbers, results show that use of Smith-van Ingen transition prediction procedure does not bring much compared to empirical correlations such as Michel's formula. On the other hand, it proved to be useful for low Reynolds numbers where Michel's formula failed. Since the $\mathrm{e}^{\mathrm{n}}$ transition prediction method is valid for both incompressible and compressible flows, its applicability ranges from low to high Reynolds numbers where empirical correlations are not valid. Another advantage is that $\mathrm{e}^{\mathrm{n}}$ transition procedure used in this study may easily be extended to three dimensional flows without much effort, where in such a case using such a method is essential since no empirical correlation is present for 3D flows.

In terms of computational time, solving for a single angle of attack varies from 3 to 8 minutes depending on the value of angle of attack. Time required increases with
increasing angle-of attack. The reasoning for this is that separated flows require much more cycles to be done and also it is necessary to use the Homotopy continuation method for largely separated flows which also requires additional computational time. However, compared to thin-layer Navier Stokes solutions, this method is much faster and therefore proves that it is useful.

The computer program presented in this study currently applies to single element airfoils with incompressible flows. In the future, modifications such as adding the ability of solving flow around multi-element airfoils, including compressibility effects or making the program suite able to handle three dimensional flows may be done. The author hopes that this study may be a reliable basis for such studies.

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## BLOCK ELIMINATION METHOD

To solve the system constructed above in sub-section 3.2.2, a technique called the block elimination method is used. The method is explained in detail in references [7],[8] and [27]. The method is analogous to the Gaussian elimination technique.

First step is to decompose the coefficient matrix $[\underset{\sim}{A}]$, into lower and upper triangular matrices as:
$[A] \cdot\{\underset{\sim}{\delta}\}=\{r\}$
$[\underset{\sim}{L}][\underset{\sim}{U}] \cdot\{\underset{\sim}{\delta}\}=\{\underset{\sim}{r}\}$

The tri-diagonal coefficient matrix can be written as:

$$
\left[\begin{array}{cccccc}
I & & & & &  \tag{A.2}\\
\Gamma_{1} & I & & & \cdots & 0
\end{array}\right)
$$

Where $I$ is the identity matrix. The solution procedure consists of two major steps: the forward sweep and the backward sweep.

## A.1. Forward Sweep

A.1.1. First Step of Forward Sweep

From equation A. 2 noting that $\Delta_{0}=A_{0}$,
$\left.\begin{array}{l}\Gamma_{1} \Delta_{0}=B_{1} \\ \vdots \\ \Gamma_{J} \Delta_{J-1}=B_{J}\end{array}\right\} \quad \Gamma_{j} \Delta_{j-1}=B_{j} \quad j=1,2, \ldots ., J$
$\left.\begin{array}{l}\Gamma_{1} C_{0}+\Delta_{1}=A_{1} \\ \Gamma_{2} C_{1}+\Delta_{2}=A_{2} \\ \vdots \\ \Gamma_{J} C_{J-1}+\Delta_{J}=A_{J}\end{array}\right\} \quad \Gamma_{j} C_{j-1}+\Delta_{j}=A_{j} \quad j=1,2, \ldots, J$

Matrices $\Gamma_{j}$ and $\Delta_{j}$ are defined as follows:

$$
\Gamma_{j}=\left[\begin{array}{ccc}
\gamma_{11} & \gamma_{12} & \gamma_{13}  \tag{A.5,6}\\
\gamma_{21} & \gamma_{22} & \gamma_{23} \\
0 & 0 & 0
\end{array}\right] \quad \Delta_{j}=\left[\begin{array}{ccc}
\alpha_{11} & \alpha_{12} & \alpha_{13} \\
\alpha_{21} & \alpha_{22} & \alpha_{23} \\
0 & -1 & -\frac{h_{j+1}}{2}
\end{array}\right]
$$

## A.1.2. Second Step of Forward Sweep

Letting:
$\{\underset{\sim}{w}\}=[\underset{\sim}{U}] \cdot\{\underset{\sim}{\delta}\}$
equation A .1 becomes:
$[\underset{\sim}{L}] \cdot\{\underset{\sim}{w}\}=\{\underset{\sim}{r}\}$
or:

$$
\left[\begin{array}{cccccccc}
I & & & & & & &  \tag{A.9}\\
\Gamma_{1} & I & & & \cdots & 0 & 0 \\
& \Gamma_{2} & I & & & & 0 \\
& & \ddots & \ddots & & & \vdots \\
& \vdots & & \Gamma_{j} & I & & \\
& 0 & & & \ddots & \ddots & & \\
& 0 & 0 & \cdots & & \ddots & \ddots & \\
& & & & & & \Gamma_{J} & I
\end{array}\right]\left\{\begin{array}{c}
w_{0} \\
w_{1} \\
\vdots \\
\vdots \\
w_{j} \\
\vdots \\
\vdots \\
w_{J}
\end{array}\right\}=\left\{\begin{array}{c}
r_{0} \\
r_{1} \\
\vdots \\
\vdots \\
r_{j} \\
\vdots \\
\vdots \\
r_{J}
\end{array}\right\}
$$

From the above expression:
$w_{0}=r_{0}$
$\Gamma_{1} w_{0}+w_{1}=r_{1}$
$\Gamma_{2} w_{1}+w_{2}=r_{2}$

$$
\begin{equation*}
w_{j}=r_{j}-\Gamma_{j} w_{j-1} \quad j=1,2, \ldots . ., J \tag{A.10}
\end{equation*}
$$

$\Gamma_{J} w_{J-1}+w_{J}=r_{J}$

All the terms on the right hand side of the above equation are known so $w_{j}$ vectors can be easily found. With this, the forward sweep is completed.

## A.2. Backward Sweep

Recalling:
$\{\underset{\sim}{w}=[U] \cdot\{\mathcal{O}\}$
or:

$$
\left[\begin{array}{ccccccccc}
\Delta_{0} & C_{0} & & & & & &  \tag{A.12}\\
& \Delta_{1} & C_{1} & & \cdots & 0 & 0 & \\
& & \ddots & \ddots & & & 0 & \\
& & & \ddots & \ddots & & \vdots & \\
& \vdots & & & \Delta_{j} & C_{j} & & \\
& 0 & & & & \ddots & \ddots & \\
& 0 & 0 & \cdots & & & \ddots & C_{J-1} \\
& & & & & & & \Delta_{J}
\end{array}\right]\left\{\begin{array}{c}
\delta_{0} \\
\delta_{1} \\
\vdots \\
\vdots \\
\delta_{j} \\
\vdots \\
\vdots \\
\delta_{J}
\end{array}\right\}=\left\{\begin{array}{c}
w_{0} \\
w_{1} \\
\vdots \\
\vdots \\
w_{j} \\
\vdots \\
\vdots \\
w_{J}
\end{array}\right\}
$$

From the above expression:
$\Delta_{J} \delta_{J}=w_{J}$
$\left.\begin{array}{l}\Delta_{J-1} \delta_{J-1}+C_{J-1} \delta_{J}=w_{J-1} \\ \vdots \\ \Delta_{0} \delta_{0}+C_{0} \delta_{1}=w_{0}\end{array}\right\} \quad \Delta_{j} \delta_{j}=w_{j}-C_{j} \delta_{j+1} \quad j=J-1, J-2, \ldots, 0$

In the above expression, everything except the $\delta_{j}$ vector is known, so it is easy to extract $\delta_{j}$. In this way, the perturbation quantities which are searched for are solved. This step completes the block elimination method.

## APPROXIMATION OF THE HILBERT INTEGRAL

Recalling the definition of the Hilbert integral:

$$
\begin{equation*}
\delta \bar{U}_{e}\left(x_{i}\right)=\frac{1}{\pi} \int_{x_{1}}^{x_{L}} G(\xi) \frac{d \xi}{\left(x_{i}-\xi\right)} \tag{B.1}
\end{equation*}
$$

Where $G(\xi)=\frac{d F}{d \xi}$, and $F=\left(\bar{U}_{e} \delta^{*}\right)_{i}$.

Where $\xi$ is a dummy variable over the integration range $x_{1} \leq \xi \leq x_{L}$ within $x_{i}$ is confined. Over each subinterval $\left(x_{k-1}, x_{k}\right), G(\xi)$ will be approximated by its midpoint value:

$$
\begin{equation*}
\int_{x_{k-1}}^{x} G(\xi) \frac{d \xi}{\left(x_{i}-\xi\right)}=G_{k-1 / 2} \ln \left[\frac{x_{i}-x_{k-1}}{x_{i}-x_{k}}\right] \tag{B.2}
\end{equation*}
$$

where:

$$
\begin{equation*}
G_{k-1 / 2}=\left(\frac{d F}{d \xi}\right)_{k-1 / 2}=\frac{F_{k}-F_{k-1}}{x_{k}-x_{k-1}} \tag{B.3}
\end{equation*}
$$

Replacing $G_{k-1 / 2}$ by its equivalent expression given by equation B.3, equation B. 2 can be written as:

$$
\begin{equation*}
\int_{x_{k-1}}^{x_{k}} G(\xi) \frac{d \xi}{\left(x_{i}-\xi\right)}=\frac{F_{k}-F_{k-1}}{x_{k}-x_{k-1}} \ln \left[\frac{x_{i}-x_{k-1}}{x_{i}-x_{k}}\right] \tag{B.4}
\end{equation*}
$$

Letting:

$$
\begin{equation*}
E_{k}=\left(x_{k}-x_{k-1}\right)^{-1} \ln \left[\frac{x_{i}-x_{k-1}}{x_{i}-x_{k}}\right] \tag{B.5}
\end{equation*}
$$

equation B. 4 can be further written as:

$$
\begin{equation*}
\int_{x_{k-1}}^{x_{k}} \frac{d F}{d \xi} \frac{d \xi}{\left(x_{i}-\xi\right)}=\left(F_{k}-F_{k-1}\right) E_{k}=E_{k}\left[\left(\bar{U}_{e} \delta^{*}\right)_{k}-\left(\bar{U}_{e} \delta^{*}\right)_{k-1}\right] \tag{B.6}
\end{equation*}
$$

Within the subinterval $[(i-1),(i+1)]$, the calculation will be performed assuming a linear variation of $G(\xi)$ as:

$$
\begin{equation*}
G(\xi)=A\left(\xi-x_{i}\right)+B \tag{B.7}
\end{equation*}
$$

Then:

$$
\begin{align*}
& G\left(\xi=x_{i-1 / 2}\right)=A\left(\frac{x_{i}+x_{i-1}}{2}-x_{i}\right)+B  \tag{B.8a}\\
& G_{i-1 / 2}=A\left(\frac{x_{i-1}-x_{i}}{2}\right)+B \tag{B.8b}
\end{align*}
$$

and in the same manner, $G_{i+1 / 2}$ can be written as:

$$
\begin{equation*}
G_{i+1 / 2}=A\left(\frac{x_{i+1}-x_{i}}{2}\right)+B \tag{B.9}
\end{equation*}
$$

Solving for $A$ and $B$ :

$$
\begin{equation*}
A=\frac{2\left(G_{i-1 / 2}-G_{i+1 / 2}\right)}{x_{i-1}-x_{i+1}} \tag{B.10}
\end{equation*}
$$

$$
\begin{equation*}
B=\frac{-G_{i+1 / 2}\left(x_{i}-x_{i-1}\right)+G_{i-1 / 2}\left(x_{i+1}-x_{i}\right)}{x_{i+1}-x_{i-1}} \tag{B.11}
\end{equation*}
$$

Hence equation B. 7 can be written as:

$$
\begin{equation*}
G(\xi)=2 \frac{\left(G_{i+1 / 2}-G_{i-1 / 2}\right)}{\left(x_{i+1}-x_{i-1}\right)}\left(\xi-x_{i}\right)+\frac{-G_{i+1 / 2}\left(x_{i}-x_{i-1}\right)+G_{i-1 / 2}\left(x_{i+1}-x_{i}\right)}{\left(x_{i+1}-x_{i-1}\right)} \tag{B.12}
\end{equation*}
$$

So the expression for the Hilbert Integral becomes:

$$
\begin{align*}
& \int_{x_{i-1}}^{x_{i+1}} G(\xi) \frac{d \xi}{\left(x_{i}-\xi\right)}=-2\left(G_{i+1 / 2}-G_{i-1 / 2}\right)-\left.G_{i+1 / 2} \frac{\left(x_{i}-x_{i-1}\right)}{\left(x_{i+1}-x_{i-1}\right)} \ln \left(x_{i}-\xi\right)\right|_{x_{i-1}} ^{x_{i+1}} \\
&-\left.G_{i-1 / 2} \frac{\left(x_{i+1}-x_{i}\right)}{\left(x_{i+1}-x_{i-1}\right)} \ln \left(x_{i}-\xi\right)\right|_{x_{i-1}} ^{x_{i+1}} \\
& \int_{x_{i-1}}^{x_{i+1}} G(\xi) \frac{d \xi}{\left(x_{i}-\xi\right)}=-2\left(G_{i+1 / 2}-G_{i-1 / 2}\right)+\left[\frac{G_{i+1 / 2}\left(x_{i}-x_{i-1}\right)+G_{i-1 / 2}\left(x_{i+1}-x_{i}\right)}{\left(x_{i+1}-x_{i-1}\right)}\right] \ln \left[\frac{x_{i}-x_{i-1}}{x_{i}-x_{i+1}}\right](\mathrm{B}
\end{align*}
$$

Now, $G$ can be expressed in terms of $d F / d \xi$ :

$$
\begin{equation*}
G_{i+1 / 2}=\left(\frac{d F}{d \xi}\right)_{i+1 / 2}=\frac{F_{i+1}-F_{i}}{x_{i+1}-x_{i}} \quad G_{i-1 / 2}=\left(\frac{d F}{d \xi}\right)_{i-1 / 2}=\frac{F_{i}-F_{i-1}}{x_{i}-x_{i-1}} \tag{B.14,15}
\end{equation*}
$$

Equation B. 13 can be written as:

$$
\begin{align*}
\int_{x_{i-1}}^{x_{i+1}} G(\xi) \frac{d \xi}{\left(x_{i}-\xi\right)}=-2\left[\frac{F_{i+1}-F_{i}}{x_{i+1}-x_{i}}-\frac{F_{i}-F_{i-1}}{x_{i}-x_{i-1}}\right] & +\frac{\left(F_{i+1}-F_{i}\right)}{\left(x_{i+1}-x_{i}\right)} \frac{\left(x_{i}-x_{i-1}\right)}{\left(x_{i+1}-x_{i-1}\right)} \ln \left|\frac{x_{i}-x_{i-1}}{x_{i}-x_{i+1}}\right|  \tag{B.16}\\
& +\frac{\left(F_{i}-F_{i-1}\right)}{\left(x_{i}-x_{i-1}\right)} \frac{\left(x_{i+1}-x_{i}\right)}{\left(x_{i+1}-x_{i-1}\right)} \ln \left|\frac{x_{i}-x_{i-1}}{x_{i}-x_{i+1}}\right|
\end{align*}
$$

Let $E_{i}^{i}=\frac{\left(x_{i+1}-x_{i}\right)}{\left(x_{i}-x_{i-1}\right)\left(x_{i+1}-x_{i-1}\right)} \ln \left|\frac{x_{i}-x_{i-1}}{x_{i}-x_{i+1}}\right|+\frac{2}{\left(x_{i}-x_{i-1}\right)}$

And $E_{i+1}^{i}=\frac{\left(x_{i}-x_{i-1}\right)}{\left(x_{i+1}-x_{i}\right)\left(x_{i+1}-x_{i-1}\right)} \ln \left|\frac{x_{i}-x_{i-1}}{x_{i}-x_{i+1}}\right|-\frac{2}{\left(x_{i+1}-x_{i}\right)}$

Therefore, Hilbert Integral can be approximated by the formula:

$$
\begin{align*}
H i=E_{2}^{i}\left(F_{2}-F_{1}\right) & +E_{3}^{i}\left(F_{3}-F_{2}\right)+\ldots \ldots \ldots .+E_{L-1}^{i}\left(F_{L-1}-F_{L-2}\right)+E_{L}^{i}\left(F_{L}-F_{L-1}\right) \\
& =-E_{2}^{i} F_{1}+\left(E_{2}^{i}-E_{3}^{i}\right) F_{2}+\ldots \ldots \ldots .+\left(E_{L-1}^{i}-E_{L}^{i}\right) F_{L-1}+E_{L}^{i} F_{L} \tag{B.19}
\end{align*}
$$

and $E_{i}^{i}=E_{L+1}^{i}=0$.

The coefficients $C_{i j}$ are now defined as:

$$
\begin{equation*}
C_{i j}=\frac{1}{\pi}\left(E_{j}^{i}-E_{j+1}^{i}\right) \quad \text { and } \quad F_{j}=\delta^{*}\left(x_{j}\right) \bar{U}_{e}\left(x_{j}\right) \tag{B.19,20}
\end{equation*}
$$

Recalling that $\bar{U}_{e_{i}}$ can be written as:
$\bar{U}_{e_{i}}=\bar{U}_{e_{i}}^{o}+\sum_{j=1}^{N} C_{i j} D_{j}$
where $C_{i j}$ as defined above and $D_{j}=\bar{U}_{e} \delta^{*}$. In this form equation B. 22 provides a boundary condition which represents the viscous/inviscid interaction. It can be generalized to the form:
$\bar{U}_{e}(x)=\bar{U}_{e}^{\kappa}(x)+\sum_{j=1}^{N} C_{i j}\left[\left(\bar{U}_{e} \delta^{*}\right)_{j}-\left(\bar{U}_{e} \delta^{*}\right)_{j}^{\kappa}\right]$
where $\bar{U}_{e}^{\kappa}(x)$ corresponds to the inviscid velocity distribution which contains the displacement thickness effect $\left(\delta^{*}\right)^{\kappa}$ coming from the previous iteration $\kappa$.

## DERIVATION OF THE ORR-SOMMERFELD EQUATION

In order to derive the stability equations, one starts from the equations of motion written in Cartesian coordinates. For viscous and incompressible flows, these are the x momentum, $y$-momentum and $z$-momentum equations, and the continuity equation. In the below equations, * stand for dimensional quantities.

X-momentum equation:
$\frac{\partial u^{*}}{\partial t^{*}}+u^{*} \frac{\partial u^{*}}{\partial x^{*}}+v^{*} \frac{\partial u^{*}}{\partial y^{*}}+w^{*} \frac{\partial u^{*}}{\partial z^{*}}=-\frac{1}{\rho^{*}} \frac{\partial p^{*}}{\partial x^{*}}=v^{*}\left(\frac{\partial^{2} u^{*}}{\partial x^{* 2}}+\frac{\partial^{2} u^{*}}{\partial y^{* 2}}+\frac{\partial^{2} u^{*}}{\partial z^{* 2}}\right)$

Y-momentum equation:

$$
\begin{equation*}
\frac{\partial v^{*}}{\partial t^{*}}+u^{*} \frac{\partial v^{*}}{\partial x^{*}}+v^{*} \frac{\partial v^{*}}{\partial y^{*}}+w^{*} \frac{\partial v^{*}}{\partial z^{*}}=-\frac{1}{\rho^{*}} \frac{\partial p^{*}}{\partial y^{*}}=v^{*}\left(\frac{\partial^{2} v^{*}}{\partial x^{* 2}}+\frac{\partial^{2} v^{*}}{\partial y^{* 2}}+\frac{\partial^{2} v^{*}}{\partial z^{* 2}}\right) \tag{C.2}
\end{equation*}
$$

Z-momentum equation:

$$
\begin{equation*}
\frac{\partial w^{*}}{\partial t^{*}}+u^{*} \frac{\partial w^{*}}{\partial x^{*}}+v^{*} \frac{\partial w^{*}}{\partial y^{*}}+w^{*} \frac{\partial w^{*}}{\partial z^{*}}=-\frac{1}{\rho^{*}} \frac{\partial p^{*}}{\partial z^{*}}=v^{*}\left(\frac{\partial^{2} w^{*}}{\partial x^{* 2}}+\frac{\partial^{2} w^{*}}{\partial y^{* 2}}+\frac{\partial^{2} w^{*}}{\partial z^{* 2}}\right) \tag{C.3}
\end{equation*}
$$

Continuity equation:

$$
\begin{equation*}
\frac{\partial u^{*}}{\partial x^{*}}+\frac{\partial v^{*}}{\partial y^{*}}+\frac{\partial w^{*}}{\partial z^{*}}=0 \tag{C.4}
\end{equation*}
$$

Dividing the flow into a steady, mean flow and unsteady small perturbations:

$$
\begin{align*}
& u^{*}=U^{*}\left(x^{*}, y^{*}, z^{*}\right)+\hat{u}^{*}\left(x^{*}, y^{*}, z^{*}, t^{*}\right)  \tag{C.5a}\\
& v^{*}=V^{*}\left(x^{*}, y^{*}, z^{*}\right)+\hat{v}^{*}\left(x^{*}, y^{*}, z^{*}, t^{*}\right) \tag{C.5b}
\end{align*}
$$

$w^{*}=W^{*}\left(x^{*}, y^{*}, z^{*}\right)+\hat{w}^{*}\left(x^{*}, y^{*}, z^{*}, t^{*}\right)$
$p^{*}=P^{*}\left(x^{*}, y^{*}, z^{*}\right)+\hat{p}^{*}\left(x^{*}, y^{*}, z^{*}, t^{*}\right)$

In the above formulation, capital lettered terms stand for steady quantities and variables with a cap stand for time dependent quantities. (e.g. $U^{*}$ is steady where $\hat{u}^{*}$ is time dependent)

The strong assumption made is the parallel flow assumption, which may be formulated as $V^{*}=0, U^{*}=U^{*}(y)$ and $W^{*}=W^{*}(y)$ only. Therefore it states that the main flow quantities are independent of stream wise distance. Even parallel flow is exactly realized for special classes of flows such as Couette and Poiseuille flows, it is at most a good assumption for the case of boundary layer flows since $\partial U^{*} / \partial x^{*} \ll \partial U^{*} / \partial y^{*}$.

Using the parallel flow assumption, dropping out the quadratic terms and further assuming that the mean flow satisfies the equations of motion, on obtains:

$$
\begin{align*}
& \frac{\partial \hat{u}^{*}}{\partial t^{*}}+U^{*} \frac{\partial \hat{u}^{*}}{\partial x^{*}}+v^{*} \frac{\partial U^{*}}{\partial y^{*}}+W^{*} \frac{\partial \hat{u}^{*}}{\partial z^{*}}=-\frac{1}{\rho^{*}} \frac{\partial \hat{p}^{*}}{\partial x^{*}}=v^{*}\left(\frac{\partial^{2} \hat{u}^{*}}{\partial x^{* 2}}+\frac{\partial^{2} \hat{u}^{*}}{\partial y^{* 2}}+\frac{\partial^{2} \hat{u}^{*}}{\partial z^{* 2}}\right)  \tag{C.6}\\
& \frac{\partial \hat{v}^{*}}{\partial t^{*}}+U^{*} \frac{\partial \hat{v}^{*}}{\partial x^{*}}+W^{*} \frac{\partial \hat{v}^{*}}{\partial z^{*}}=-\frac{1}{\rho^{*}} \frac{\partial \hat{p}^{*}}{\partial y^{*}}=v^{*}\left(\frac{\partial^{2} \hat{v}^{*}}{\partial x^{* 2}}+\frac{\partial^{2} \hat{v}^{*}}{\partial y^{* 2}}+\frac{\partial^{2} \hat{v}^{*}}{\partial z^{* 2}}\right)  \tag{C.7}\\
& \frac{\partial \hat{w}^{*}}{\partial t^{*}}+U^{*} \frac{\partial \hat{w}^{*}}{\partial x^{*}}+\hat{v}^{*} \frac{\partial \hat{w}^{*}}{\partial y^{*}}+W^{*} \frac{\partial \hat{w}^{*}}{\partial z^{*}}=-\frac{1}{\rho^{*}} \frac{\partial \hat{p}^{*}}{\partial z^{*}}=v^{*}\left(\frac{\partial^{2} \hat{w}^{*}}{\partial x^{* 2}}+\frac{\partial^{2} \hat{w}^{*}}{\partial y^{* 2}}+\frac{\partial^{2} \hat{w}^{*}}{\partial z^{* 2}}\right)  \tag{C.8}\\
& \frac{\partial \hat{u}^{*}}{\partial x^{*}}+\frac{\partial \hat{v}^{*}}{\partial y^{*}}+\frac{\partial \hat{w}^{*}}{\partial z^{*}}=0 \tag{C.9}
\end{align*}
$$

Here, it is assumed that the disturbances are small (small disturbance theory) so that the quadratic terms are neglected. (e.g. $\hat{u}^{*} \partial \hat{u}^{*} / \partial x^{*} \ll U^{*} \partial \hat{u}^{*} / \partial x^{*}$, etc.)

Non-dimensionalizing velocities with $U_{R}^{*}$, lengths by $L_{R}^{*}$ and pressure terms by $\rho^{*} U_{R}^{* 2}$ one obtains:

X-momentum equation:
$\frac{\partial \hat{u}}{\partial t}+U \frac{\partial \hat{u}}{\partial x}+W \frac{\partial \hat{u}}{\partial z}+\hat{v} \frac{d U}{d y}=-\frac{\partial \hat{p}}{\partial x}-\frac{1}{R}\left(\frac{\partial^{2} \hat{u}}{\partial x^{2}}+\frac{\partial^{2} \hat{u}}{\partial y^{2}}+\frac{\partial^{2} \hat{u}}{\partial z^{2}}\right)$

Y-momentum equation:
$\frac{\partial \hat{v}}{\partial t}+U \frac{\partial \hat{v}}{\partial x}+W \frac{\partial \hat{v}}{\partial z}=-\frac{\partial \hat{p}}{\partial y}-\frac{1}{R}\left(\frac{\partial^{2} \hat{v}}{\partial x^{2}}+\frac{\partial^{2} \hat{v}}{\partial y^{2}}+\frac{\partial^{2} \hat{v}}{\partial z^{2}}\right)$

Z-momentum equation:
$\frac{\partial \hat{w}}{\partial t}+U \frac{\partial \hat{w}}{\partial x}+W \frac{\partial \hat{w}}{\partial z}+\hat{v} \frac{d W}{d y}=-\frac{\partial \hat{p}}{\partial z}-\frac{1}{R}\left(\frac{\partial^{2} \hat{w}}{\partial x^{2}}+\frac{\partial^{2} \hat{w}}{\partial y^{2}}+\frac{\partial^{2} \hat{w}}{\partial z^{2}}\right)$

Continuity equation:
$\frac{\partial \hat{u}}{\partial x}+\frac{\partial \hat{v}}{\partial y}+\frac{\partial \hat{w}}{\partial z}=0$

Where the Reynolds number is defined as:
$R=\frac{\rho^{*} U_{R}^{*} L_{R}^{*}}{\mu^{*}}$

The mean laminar flow in x-direction is assumed to be influenced by a disturbance composed of a number of discrete partial fluctuations each of which is assumed to consist of a wave propagating in x -direction.
$\hat{u}=\bar{u}(y) e^{i(\alpha x+\beta z-\omega t)}$
$\hat{v}=\bar{v}(y) e^{i(\alpha x+\beta z-\omega t)}$
$\hat{w}=\bar{w}(y) e^{i(\alpha x+\beta z-\omega t)}$
$\hat{p}=\bar{p}(y) e^{i(\alpha x+\beta z-\omega t)}$

In the above equation C.15, $\bar{u}(y), \bar{v}(y), \bar{w}(y)$ and $\bar{p}(y)$ are the disturbance amplitude functions with:
$\alpha:$ Wave number in x -direction and $\alpha=2 \pi / \lambda$ with $\lambda$ being the wavelength.
$\beta$ : Wave number is z-direction.
$\omega$ : Circular frequency (complex) as $\omega=\omega_{r}+i \omega_{i}$.

The term $\omega_{r}$ in the above definition is the frequency and $\omega_{i}$ is the amplification rate. Substituting the above relations into equations of motion, one obtains equations C.16-19 which are given below with primes denoting $\partial / \partial y$ or $d / d y$.

X-momentum equation:

$$
\begin{equation*}
-i \omega \bar{u}+U i \alpha \bar{u}+W i \beta \bar{u}+\bar{v} U^{\prime}=-i \alpha \bar{p}+1 / R\left(-\alpha^{2} \bar{u}+\bar{u}^{\prime \prime}-\beta^{2} \bar{u}\right) \tag{C.16}
\end{equation*}
$$

Y-momentum equation:
$-i \omega \bar{v}+U i \alpha \bar{v}+W i \beta \bar{v}=\bar{p}+1 / R\left(-\alpha^{2} \bar{v}+\bar{v}^{\prime \prime}-\beta^{2} \bar{v}\right)$

Z-momentum equation:

$$
\begin{equation*}
-i \omega \bar{w}+U i \alpha \bar{w}+W i \beta \bar{w}=-i \beta \bar{p}+1 / R\left(-\alpha^{2} \bar{w}+\bar{w}^{\prime \prime}-\beta^{2} \bar{w}\right) \tag{C.18}
\end{equation*}
$$

Continuity equation:

$$
\begin{equation*}
i \alpha \bar{u}+\vec{v}+i \beta \bar{w}=0 \tag{C.19}
\end{equation*}
$$

Now with $D$ denoting $\partial / \partial y$ or $d / d y$ and rearranging terms, one obtainsthe following system of equations:

$$
\begin{align*}
& D U \bar{v}+i(\alpha U+\beta W-\omega) \bar{u}=-i \alpha \bar{p}+1 / R\left[D^{2}-\left(\alpha^{2}+\beta^{2}\right)\right] \bar{u}  \tag{C.20}\\
& \quad i(\alpha U+\beta W-\omega) \bar{v}=-\bar{p}^{\prime}+1 / R\left[D^{2}-\left(\alpha^{2}+\beta^{2}\right)\right] \bar{v}  \tag{C.21}\\
& D W \bar{v}+i(\alpha U+\beta W-\omega) \bar{w}=-i \beta \bar{p}+1 / R\left[D^{2}-\left(\alpha^{2}+\beta^{2}\right)\right] \bar{w}  \tag{C.22}\\
& i(\alpha \bar{u}+\beta \bar{w})+D \bar{v}=0 \tag{C.23}
\end{align*}
$$

The boundary conditions for the above system of equations are given below in equation C. 24 .

$$
\begin{array}{ll}
\bar{u}(0)=\bar{v}(0)=\bar{w}(0)=0 & \text { at } \quad y=0 \text { (no slip) } \\
\bar{u} \rightarrow 0, \bar{v} \rightarrow 0, \bar{w} \rightarrow 0 & \text { as } \quad y \rightarrow \infty \text { (free stream) } \tag{C.24b}
\end{array}
$$

## C.1. Reduction to a fourth order system

Multiplying the x -momentum equation by $\alpha$, and z -momentum equation by $\beta$, and adding them, one obtains the following equation after rearranging.

$$
\begin{align*}
& i(\alpha U+\beta W-\omega)(\alpha \bar{u}+\beta \bar{w})+(\alpha D U+\beta D W) \bar{v}=-i\left(\alpha^{2}+\beta^{2}\right) \bar{p} \\
&+1 / R\left[D^{2}-\left(\alpha^{2}+\beta^{2}\right)\right](\alpha \bar{u}+\beta \bar{w}) \tag{C.25}
\end{align*}
$$

Multiplying the x -momentum equation by $\beta$ and subtracting the resultant from z momentum equation multiplied with $\alpha$, one obtains equation C.26.

$$
\begin{equation*}
i(\alpha U+\beta W-\omega)(\alpha \bar{w}-\beta \bar{u})+(\alpha D W-\beta D U) \bar{v}=1 / R\left[D^{2}-\left(\alpha^{2}+\beta^{2}\right)\right](\alpha \bar{w}-\beta \bar{u}) \tag{C.26}
\end{equation*}
$$

After these steps, the new equation system becomes:

$$
\begin{align*}
& \begin{aligned}
i(\alpha U+\beta W-\omega)(\alpha \bar{u}+\beta \bar{w})+ & (\alpha D U+\beta D W) \bar{v}=-i\left(\alpha^{2}+\beta^{2}\right) \bar{p} \\
& +1 / R\left[D^{2}-\left(\alpha^{2}+\beta^{2}\right)\right](\alpha \bar{u}+\beta \bar{w})
\end{aligned} \\
& i\left(\alpha U \_\beta W-\omega\right) \bar{v}=-\bar{p}^{\prime}+1 / R\left[D^{2}-\left(\alpha^{2}+\beta^{2}\right)\right] \bar{v}  \tag{C.25}\\
& i(\alpha U+\beta W-\omega)(\alpha \bar{w}-\beta \bar{u})+(\alpha D W-\beta D U) \bar{v}=1 / R\left[D^{2}-\left(\alpha^{2}+\beta^{2}\right)\right](\alpha \bar{w}-\beta \bar{u})  \tag{C.27}\\
& i(\alpha \bar{u}+\beta \bar{w})+D \bar{v}=0 \tag{C.26}
\end{align*}
$$

The dependent variables or in other words, the eigenfunctions of the above system of equations are: $\alpha \bar{u}+\beta \bar{w}, \bar{v}, \bar{p}$ and $\alpha D \bar{u}+\beta D \bar{w}$. Above equations C.23, C.25, C. 26 and C. 27 constitute a fourth order system. The variable $(\alpha \bar{w}-\beta \bar{u})$ appears only in equation C.26. Therefore, the eigenvalues $\alpha, \beta, \omega$ and $R$ can be determined from this fourt order system. If one needs to solve the eigenfunctions $\bar{u}$ and $\bar{w}$, equation C. 26 must be solved as well. If $\alpha$ and $\beta$ are real, equation C. 25 is the momentum equation in the direction of wave motion whereas equation C. 26 is the momentum equation normal to the direction of wave motion in the x -z plane.

## C.2. Special Form of the Stability Equations: Orr-Sommerfeld Equation

A single fourth order equation is obtained from equations C.23, C. 25, C. 26 and C. 27 by eliminating $\alpha \bar{u}+\beta \bar{w}$ from equation C. 25 by equation C. 23 , and, after differentiation with respect to y , eliminating $D \bar{p}$ by equation C.27. The resultant equation is:

$$
\begin{equation*}
\left[D^{2}-\left(\alpha^{2}+\beta^{2}\right)\right]^{2} \bar{v}=i R\left\{(\alpha U+\beta W-\omega)\left[D^{2}-\left(\alpha^{2}+\beta^{2}\right)\right]-\left(\alpha D^{2} U+\beta D^{2} W\right)\right\} \bar{v} \tag{C.28}
\end{equation*}
$$

With the boundary conditions given below in equation C.29.

$$
\begin{array}{ll}
\bar{v}(0)=D \bar{v}(0)=0 & \text { at } \quad y=0 \text { (no slip) } \\
\bar{v} \rightarrow 0, D \bar{v} \rightarrow 0 & \text { as } \quad y \rightarrow \infty \text { (free stream) }
\end{array}
$$

For a two dimensional mean flow, $W=0$ and hence $\beta=0$. Therefore, equation C. 28 reduces to equation C. 30 .

$$
\begin{equation*}
\left(D^{2}-\alpha^{2}\right) \bar{v}=i R\left[(\alpha U-\omega)\left(D^{2}-\alpha^{2}\right)-\alpha^{2} D^{2} U\right] \bar{v} \tag{C.30}
\end{equation*}
$$

This equation is called the Orr-Sommerfeld equation and it is valid for a two dimensional wave in a two dimensional boundary layer.

With primes rather than the letter $D$ denoting differentiation in the normal direction, and $\phi(y) \equiv\left(\phi_{r}+i \phi_{i}\right)$ representing the complex amplitude of the disturbance stream function $\psi^{\prime}$ defined by the fluctuating velocity components $u^{\prime}$ and $v^{\prime}$ defined as shown in equation 4.1 with definition given in equation C.31, the Orr-Sommerfeld equation can be rewritten in another form given below in equation 1.2.

$$
\begin{array}{ll}
u^{\prime}=\frac{\partial \psi^{\prime}}{\partial y} & v^{\prime}=-\frac{\partial \psi^{\prime}}{\partial x} \\
\psi^{\prime}=\phi(y) \exp [i(\alpha x-\omega t)] & \tag{C.31}
\end{array}
$$

Note that in the below form of the O-S equation, the velocity component $U$ are shown with small case letter $u$ for the sake of convenience.

$$
\begin{equation*}
\left(\phi^{i v}-2 \alpha^{2} \phi^{\prime \prime}+\alpha^{4} \phi\right)=i R\left[(\alpha u-\omega)\left(\phi^{\prime \prime}-\alpha^{2} \phi\right)-\alpha u^{\prime \prime} \phi\right] \tag{1.2}
\end{equation*}
$$

## DESCRIPTION OF THE COMPUTER PROGRAM

In order to conduct this study, a computer program written before in reference [1] has been utilized. The Smith-van Ingen $e^{n}$ procedure is implemented through another one whish has been written in Fortan 77 programming language. The theory and procedure is thoroughly explained in the above chapters $2,3,4$ and 5 . In order to prepare the inputs for the $e^{\mathrm{n}}$ code, the original interactive boundary layer code is altered so that it now prints out the boundary layer grid information which is used by the $\mathrm{e}^{\mathrm{n}}$ code.

The program suite which is used in this study consists of three program codes, from which two are essentially the same except for the input and output files are different to examine upper and lower surfaces of the airfoil in consideration. The first program written originally by Dr. Serkan Özgen during his Master of Science studies in 1994 is the interactive boundary layer code with wake effects included is called the IBLW. In order to solve for the stability equations and to employ the $\mathrm{e}^{\mathrm{n}}$ procedure, a stability transition code is written which originally inputs a single velocity profile and solves for the Falkner-Skan profiles which are used to prove the feasibility of the procedure. The other two are altered versions of the stability transition code called as ENLO and ENUP, where the only difference between the two are the input and output files as mentioned above. These altered versions read the velocity profiles stored at each boundary layer grid station which are prepared by the IBLW code. The EN code can either be used to solve for the neutral stability curve, or to predict transition.

In order to analyze an airfoil at any given angle of attack, user runs the IBLW code with laminar region as long as feasible, and then runs the ENLO and ENUP code for the upper and lower surfaces of the airfoil separating from the stagnation point which is also calculated by the IBLW program extending through the trailing edge. The codes ENLO and ENUP provide the user the physical frequencies, and the amplification rates related to these frequencies. Depending on the user's choice, at a value where the original instability is amplified by a factor between $e^{8}$ to $e^{10}$, the transition location is determined and fed back into the IBLW code as an external input. The actual solution is then obtained.

Note that as the compressibility effects are neglected, the solutions are valid only within the incompressible range typically where the Mach number is less than 0.3 . Therefore the solution is limited to low to moderate Reynolds numbers. Also the Smith-van Ingen $\mathrm{e}^{\mathrm{n}}$ procedure is feasible for low to moderate angles-of-attack. For high values, laminar separation point must be used as the transition mechanism for flows with adverse pressure gradient.

The EN code has a MAIN routine and four subroutines which are briefly described below.

In MAIN routine, the number of stations, their values, and the initial station for starting the calculations are inputted. Also the user selects whether neutral stability curve or transition is required. If transition is required, number of lines to be followed is entered. This is done by giving a value for the mode flag as 0 for neutral stability of any integer less than or equal to 20 for transition prediction. Also the non-dimensional station coordinates; distances along the surface starting from the stagnation point and the external velocity distribution at each station are input. With the additional information about the chord Reynolds number, the stability Reynolds number for each station is computed. One other input parameter is the initial guesses for eigenvalues $\alpha$ and $\omega$ to be used at initiation. One may find good estimates from the Falkner-Skan solutions after determining the pressure gradient.

The velocity profiles $u$ and $u^{\prime \prime}$ at stations are read by the subroutine PROFILE, the neutral stability calculations are done by subroutine NEWTON, where the amplification rated are calculated by the subroutine AMPL.

The subroutine OS is used to solve for the Orr-Sommerfeld equation, and includes the block elimination algorithm described above. In the NEWTON subroutine, the eigenvalue procedure explained is done. After convergence is achieved, physical frequencies are computed. The AMPL subroutine is used for calculating the amplification factor which is the key value in determining the onset of transition. After the code is run, the amplification factors are printed at all stations, where the user is responsible to curvefit the data and to probe the location where the amplification rate has reached the value desired to set the transition point.


[^0]:    Keywords : Boundary Layer Flows, Transition Prediction, Blowing Velocity, Airfoil Performance, Interactive Method

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